

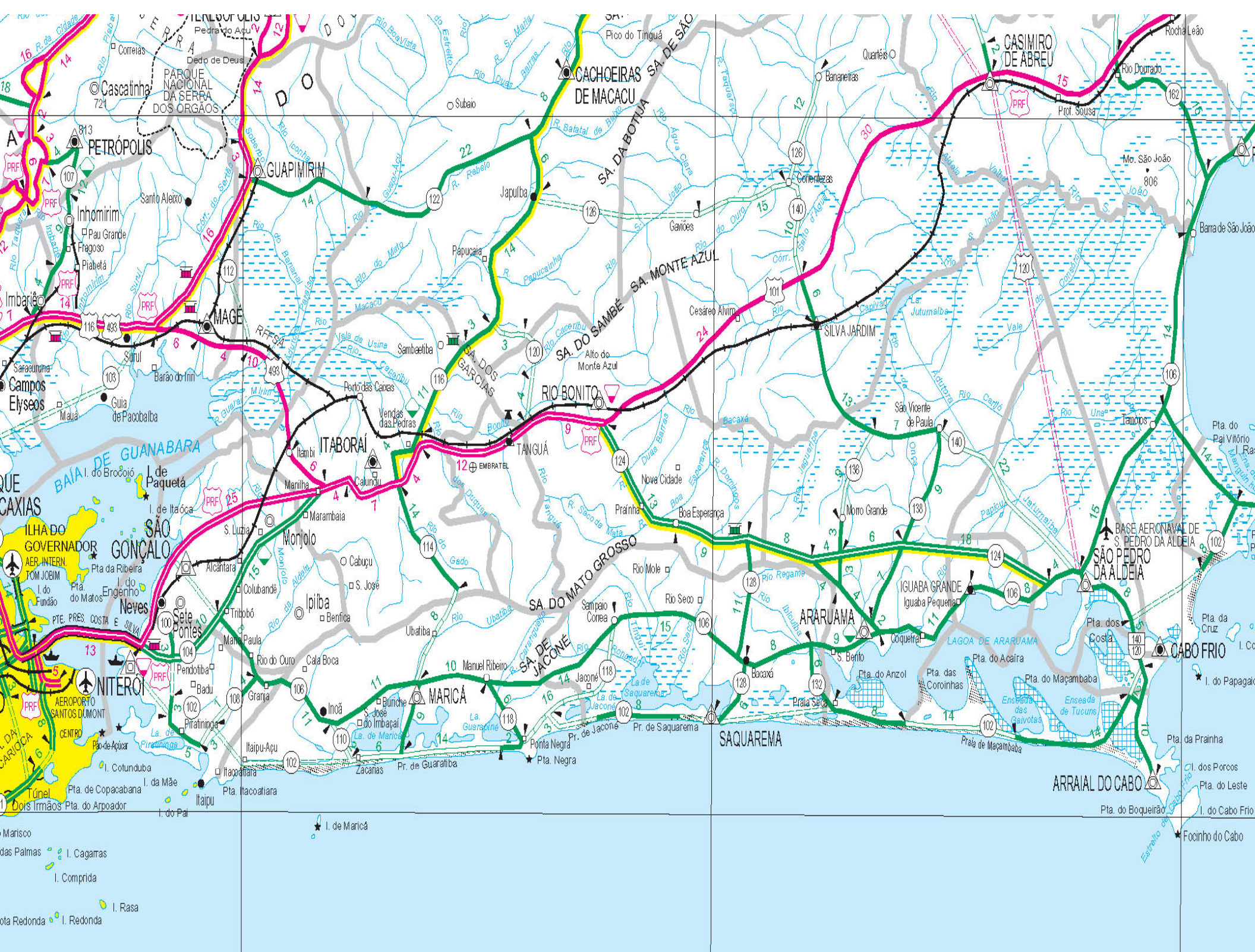
INNOVATIVE CONCEPTS FOR WTE AND NATURAL GAS TURBINE POWER PLANTS – AN ECONOMICALLY FEASIBLE SOLUTION FOR RIO DE JANEIRO.

Dr. Sergio Guerreiro

FINAL DESTINATION OF WASTE IN RIO GRAMACHO LANDFILL → 8,000 TON/D

WILL CLOSE SOON

**PROBLEM → NEW PACIENCIA LANDFILL
WAS CANCELLED DUE TO DIFFICULTIES
WITH ENVIRONMENTAL LICENSE**



USUAL WTE DIFFICULTIES

- 1. HIGH INVESTMENT***
- 2. BOILER CORROSION <-> LOW EFFICIENCY***
- 3. HIGH O&M COSTS***
- 3. LOW ENERGY / TON OF MSW***
- 4. LOW TIP FEE IN MANY PLACES (BRAZIL)***
- 5. CARBON CREDITS CAN HELP***

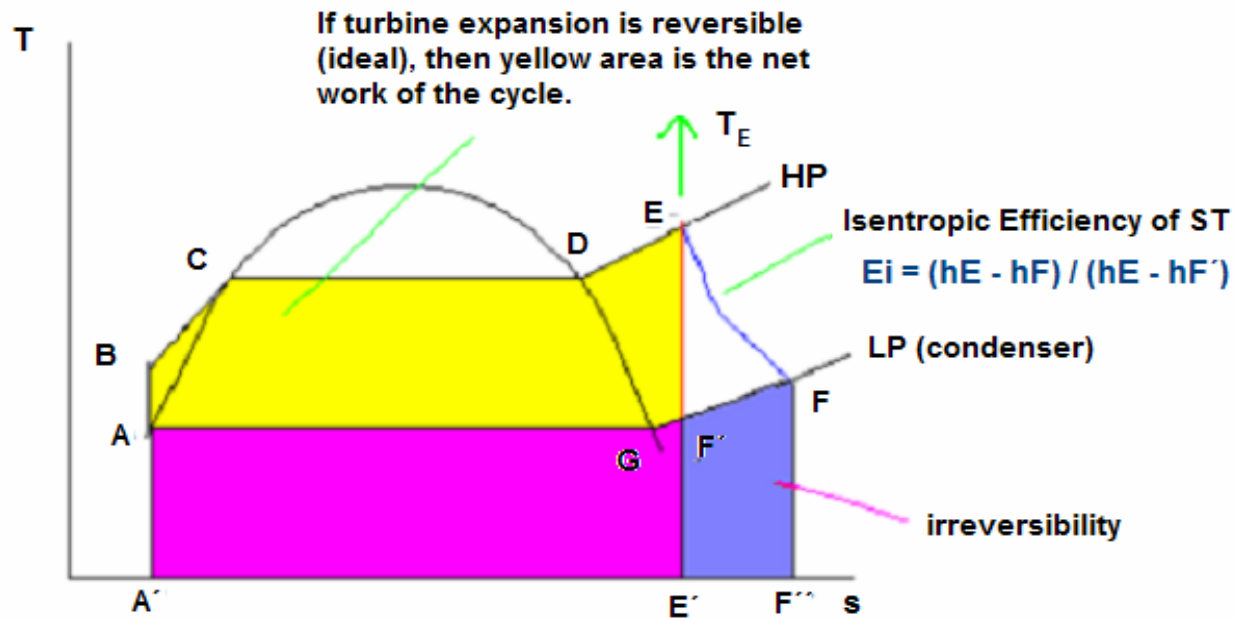
ADVANTAGES OF WTE

- 1. ONLY VIABLE ALTERNATIVE TO LANDFILLS**
- 2. ENVIRONMENTALLY SUSTAINABLE**
- 3. MATERIALS AND ENERGY RECOVERY**
- 4. CAN BE COMPLETELY CONTROLLED**
- 5. WE CAN AND SHOULD MONITOR EMISSIONS**

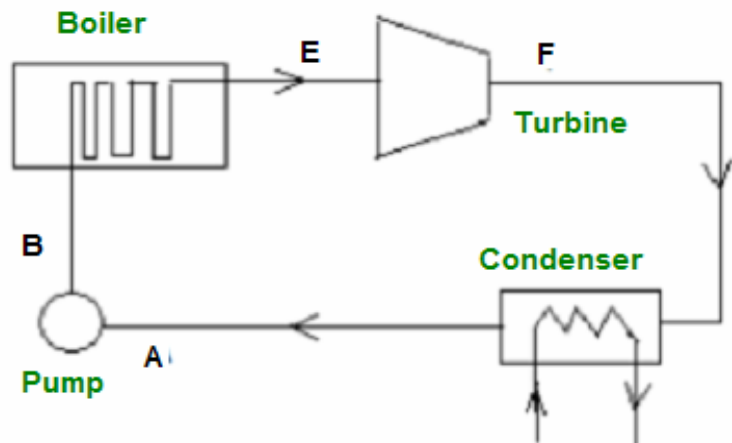
STEAM CYCLE EFFICIENCY

$$W(\text{net}) = Q(\text{in}) - Q(\text{out}) = (A' - B - C - D - E - E' - A') - (A' - A - G - F - F' - A') = (\text{blue} + \text{red}) - (\text{green} + \text{blue}) = \text{red} - \text{green}$$

$$dQ = T ds$$

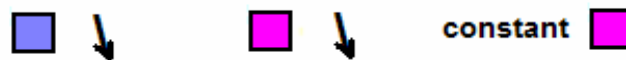


$$\text{Efficiency} = W(\text{net}) / Q(\text{in}) = (\text{yellow} - \text{blue}) / (\text{pink} + \text{yellow})$$



TO INCREASE EFFICIENCY

$$Q_{in} = \text{constant} = (\text{pink box} + \text{yellow box}) = \text{Fuel}$$

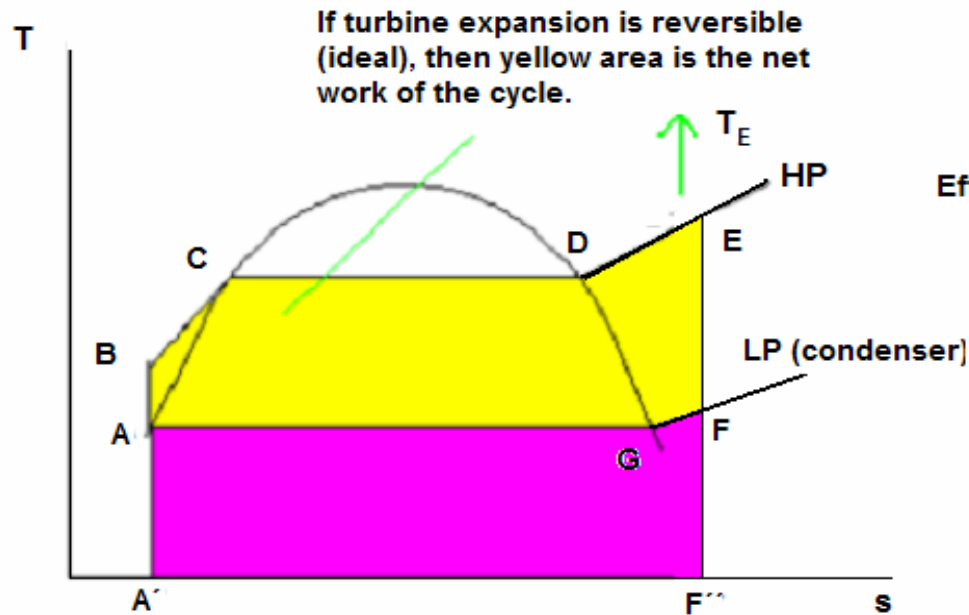


HP $\uparrow$ T_E $\uparrow$ superheater
LP $\downarrow$ condenser

STEAM CYCLE EFFICIENCY (IDEAL)

$$W(\text{net}) = Q(\text{in}) - Q(\text{out}) = (A' - B - C - D - E - F'' - A') - (A' - A - G - F - F'' - A') = (\text{pink} + \text{yellow}) - (\text{pink}) = \text{yellow}$$

$$dQ = T ds$$

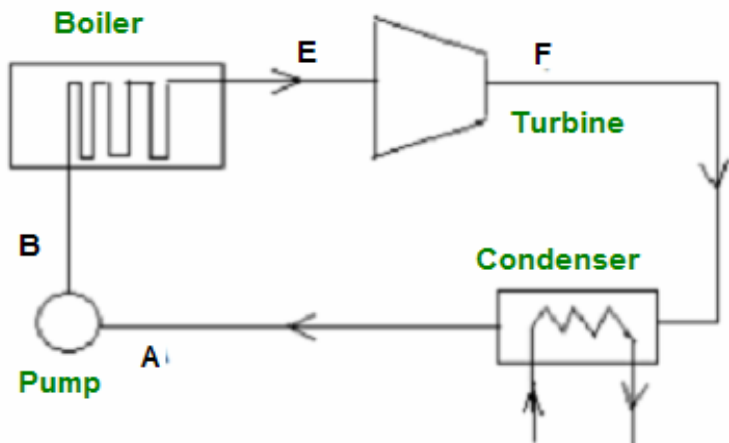


$$\text{Efficiency} = W(\text{net}) / Q(\text{in}) = (\text{yellow box}) / (\text{pink box} + \text{yellow box})$$

TO INCREASE EFFICIENCY

1) $Q_{in} = \text{constant} = (\text{pink box} + \text{yellow box}) = \text{Fuel}$

condenser



2) constant

HP $\uparrow$ T_E $\uparrow$
superheater



WtE Facility Bilbao, Spain



10/07/2004



EXEMPLOS – INCINERAÇÃO ZABALGARBI, ESPANHA

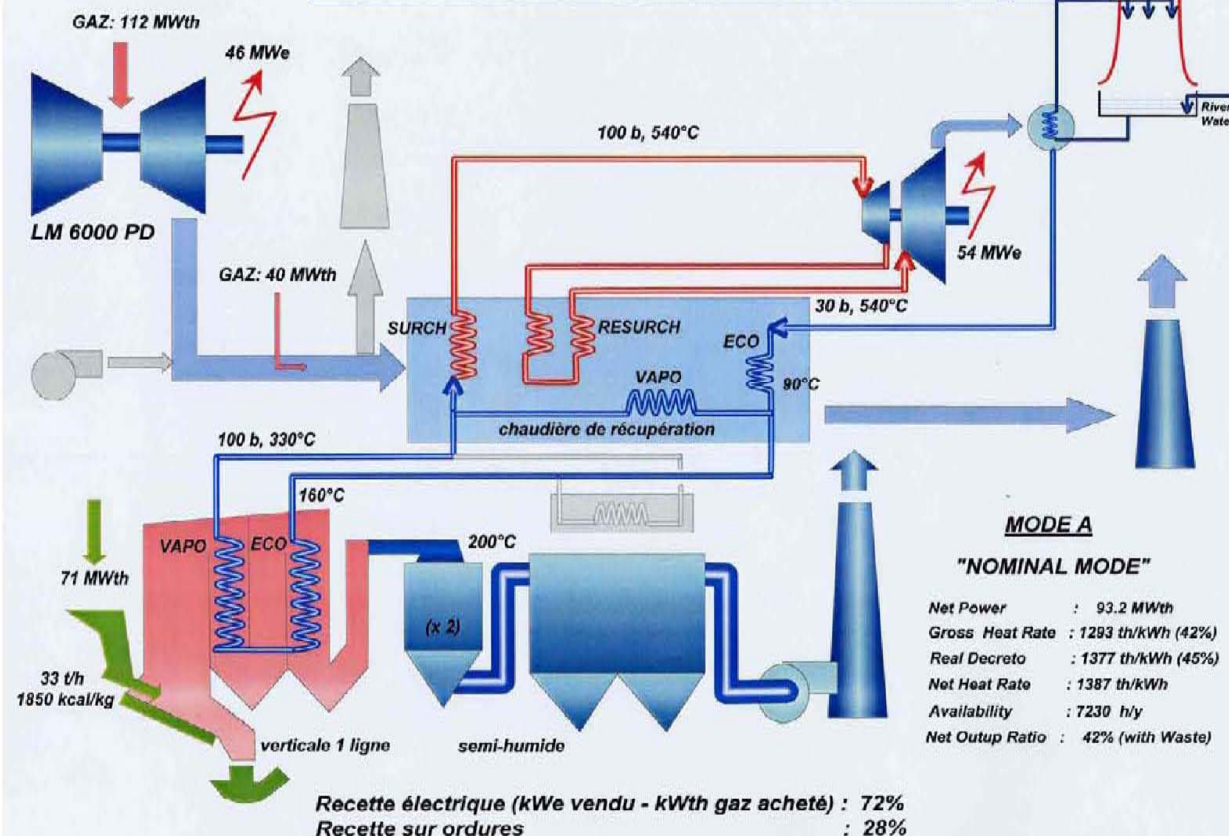


MARTIN GmbH

für Umwelt- und Energietechnik

Combined WtE / Natural Gas Turbine

- CNIM Bilbao Concept -



10/07/2004

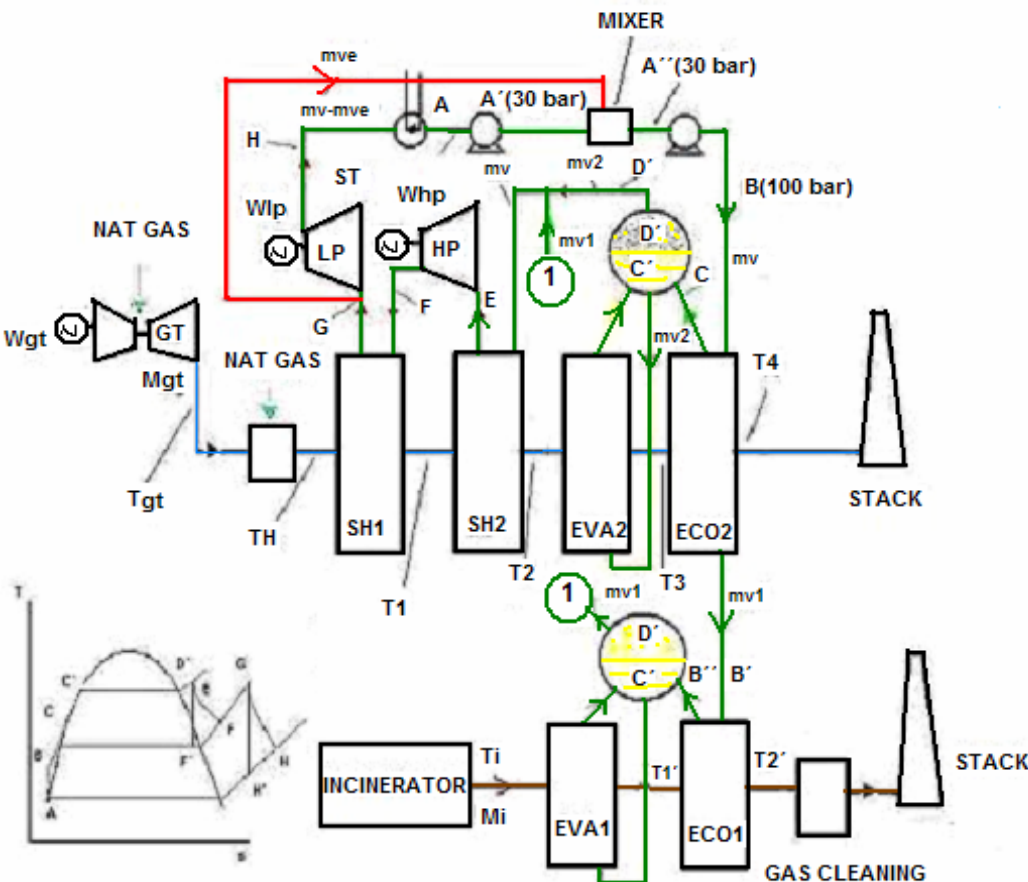


Waste-to-Energy Research and Technology Council

Fall Meeting Columbia University, Oct.7-8, 2004

60





STEAM CYCLE EFFICIENCY = 36.1%

ST POWER = $144.2 \times 0.361 = 52.06$ MWe

TOTAL POWER = $46 + 52.06 = 98.06$ MWe

OVERALL EFF. = $98.06 / (152 + 71) = 43.97\%$

PAID FUEL EFF. = $98.06 / 152 = 64.51\%$

NG EFF. = $76.76 / 152 = 50.50\%$

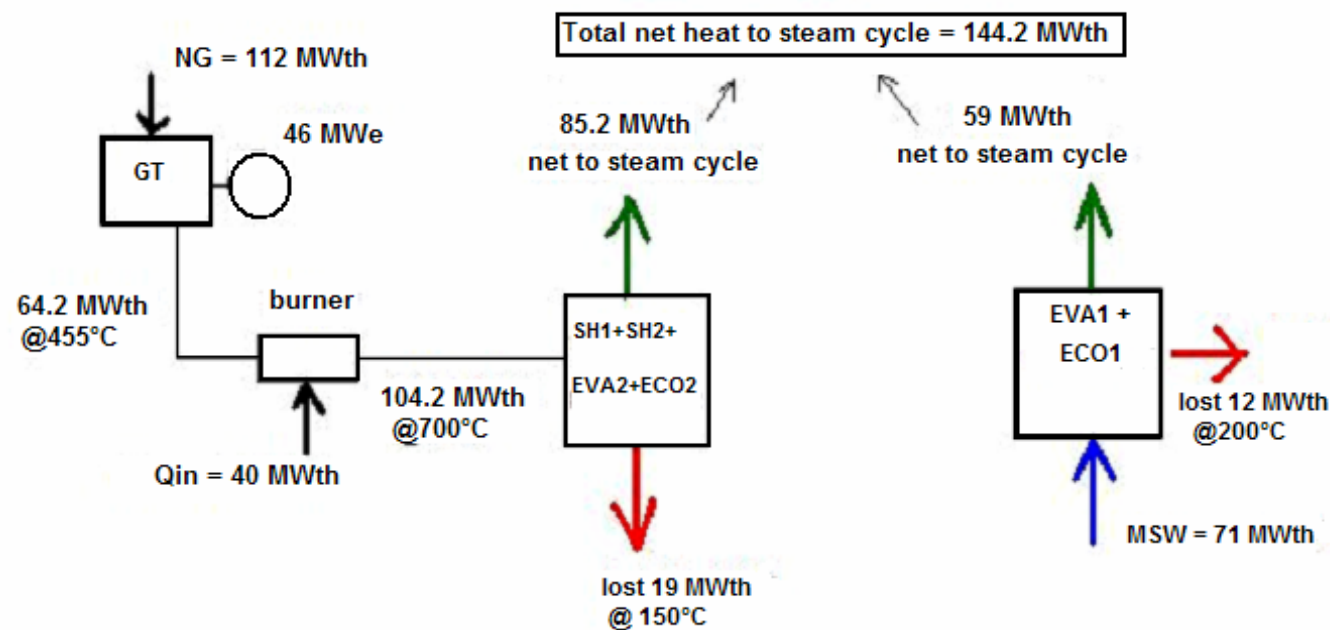
WASTE EFF. = $21.3 / 71 = 30\%$

WASTE CONTRIBUTION

$21.30 / 98.06 = 21.72\%$ OF TOTAL ENERGY

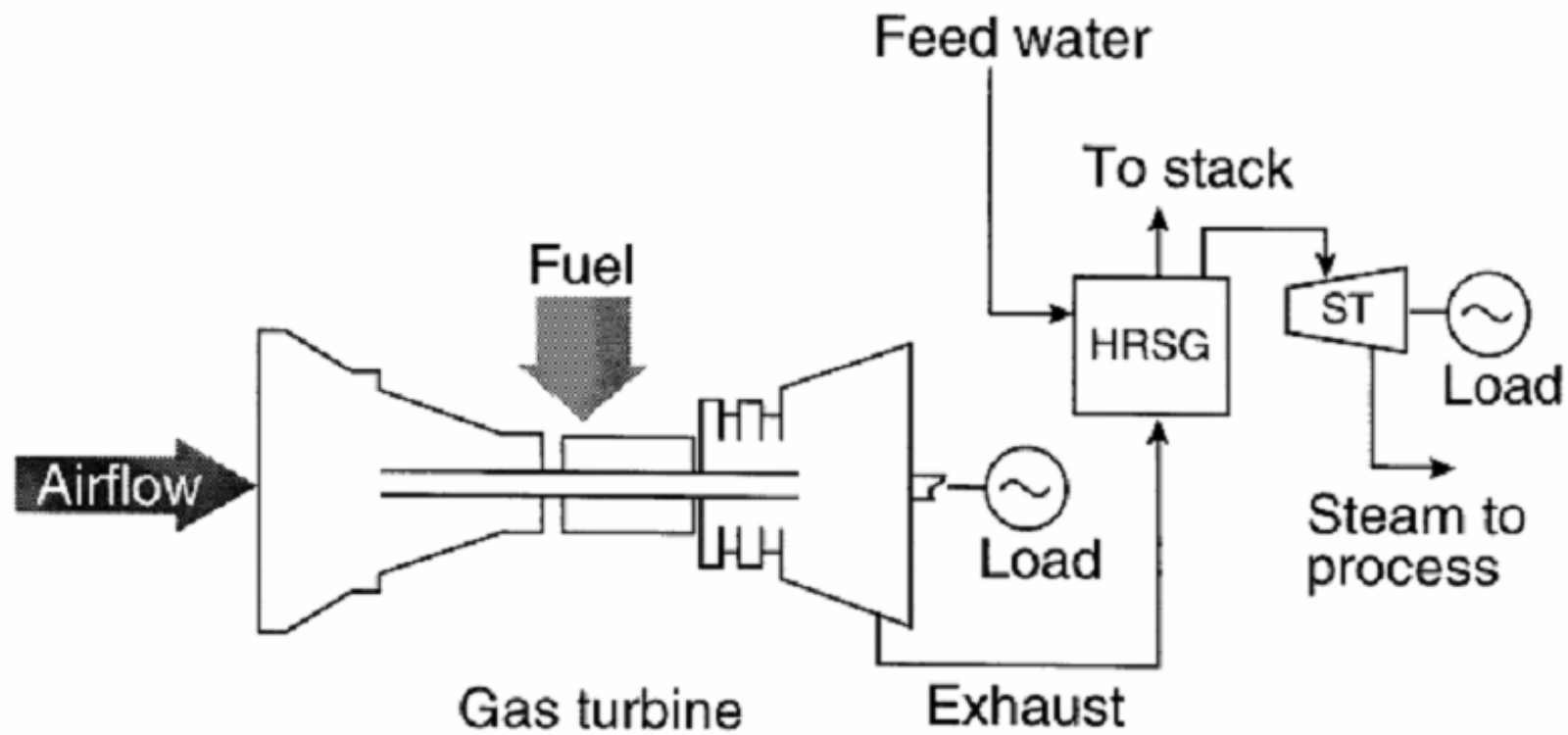
MWth / TON = $21.3 / 33 = 0.64$ (1,850 Kcal/Kg)

MWth / TON = $21.3 / 33 = 0.84$ (2,400 Kcal/Kg)

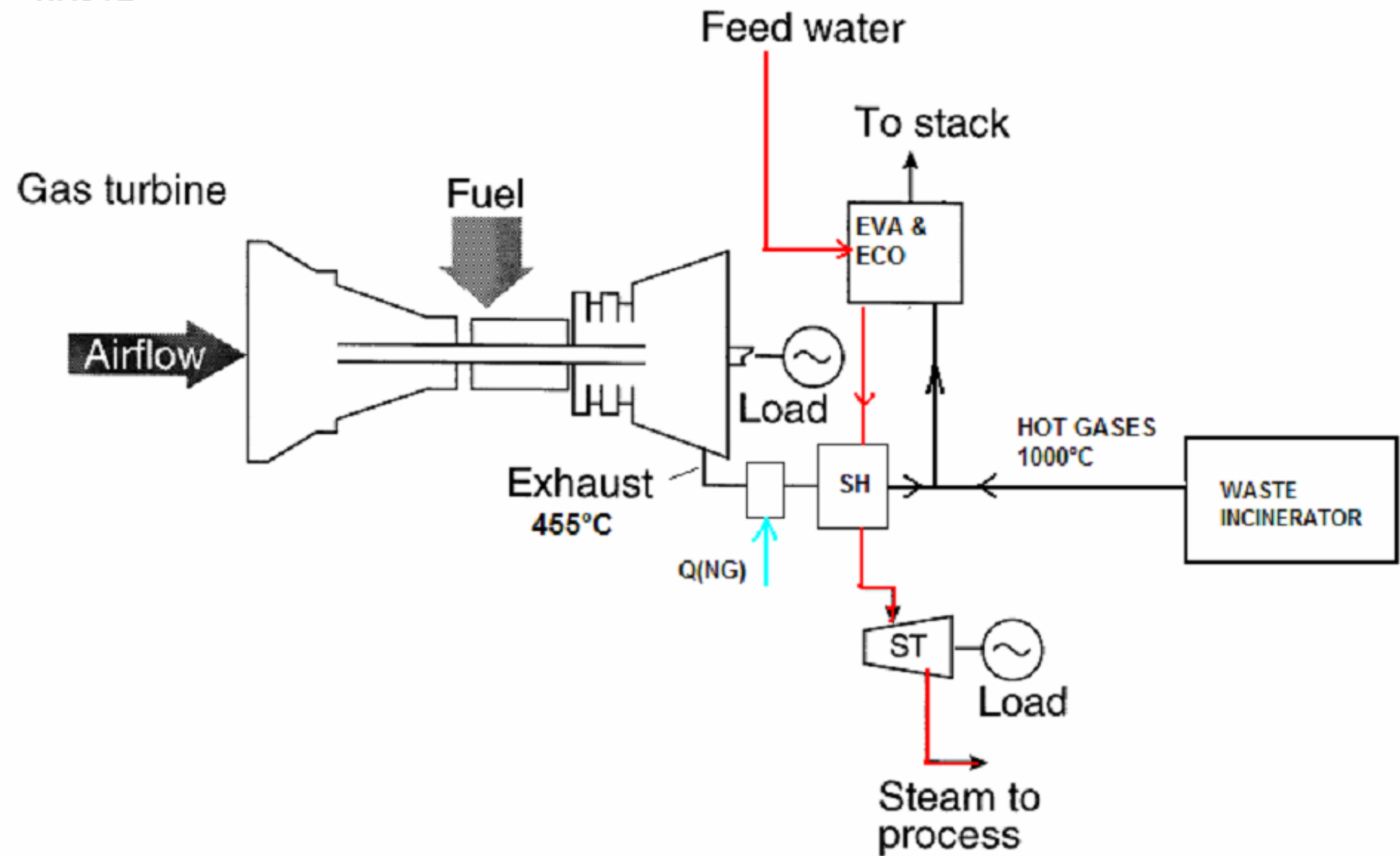


**ZABALGARBI
ORIGINAL**

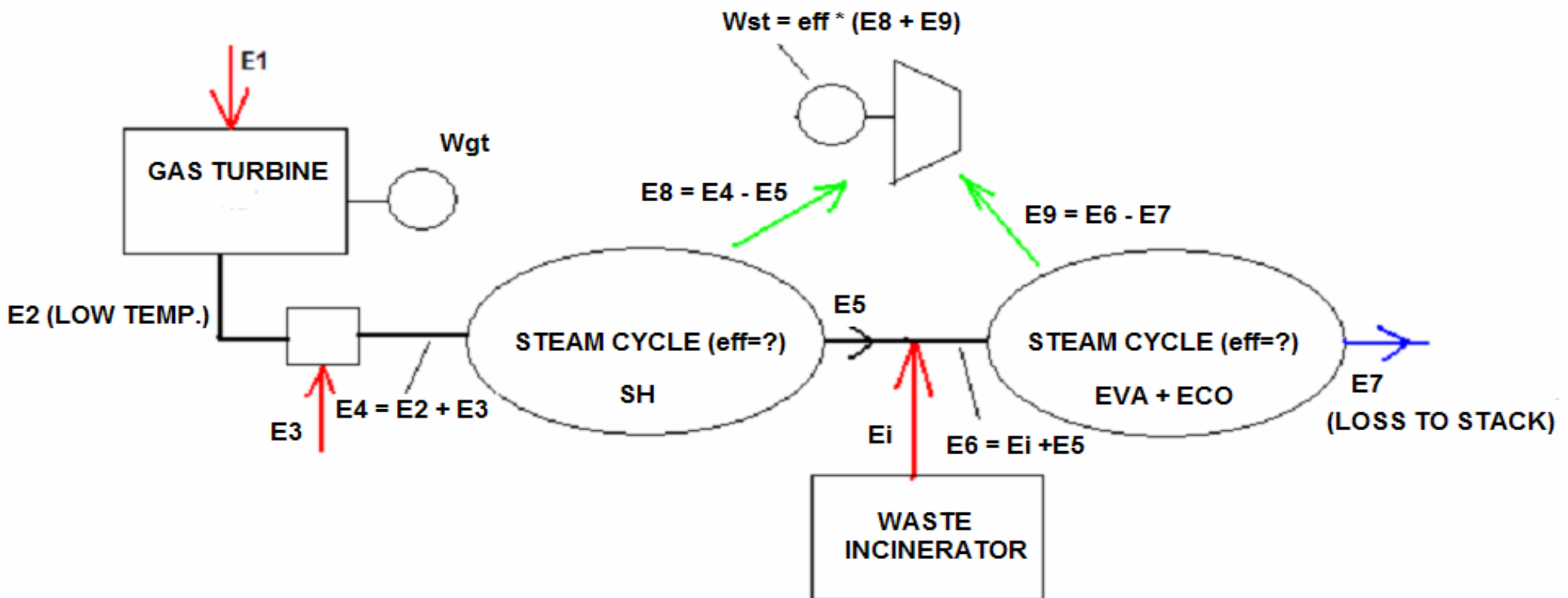
GAS TURBINE WITH STANDARD COMBINED CYCLE



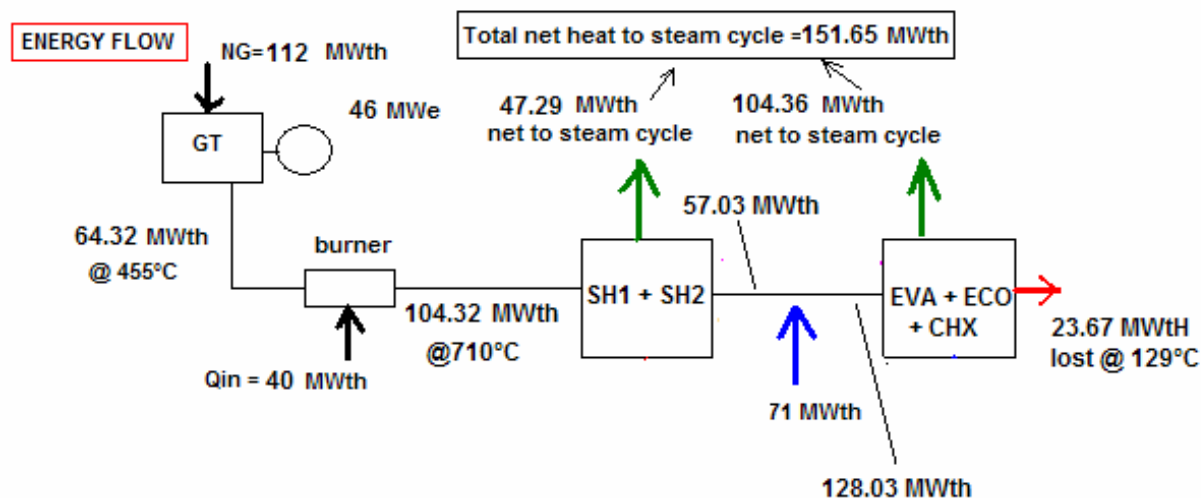
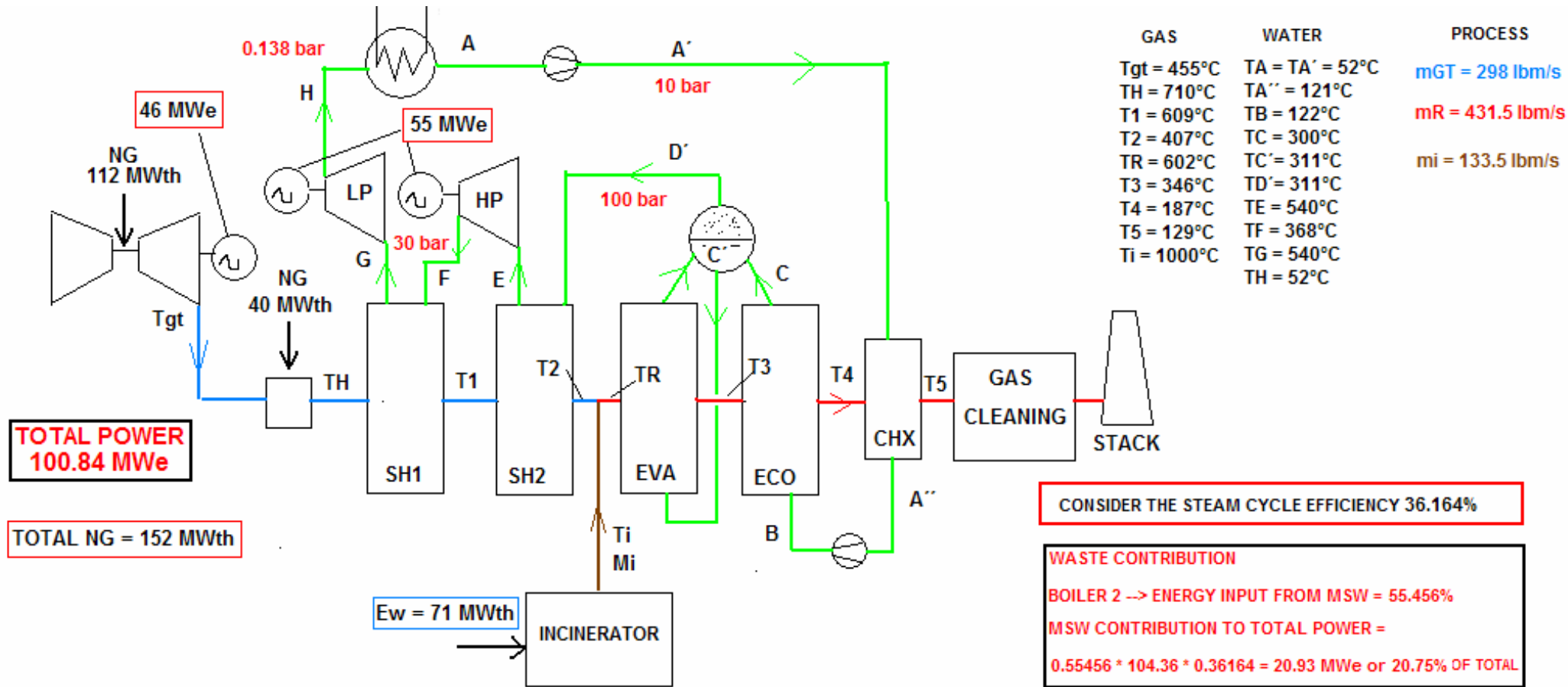
**GAS TURBINE WITH
SPECIAL COMBINED
CYCLE INCLUDING
WASTE**



ENERGY FLOW FOR WASTE / NG SYSTEMS



1. WHEN WE "HANG" THE WTE PLANT BETWEEN THE SUPERHEATER AND THE EVAPORATOR WE AVOID CORROSION IN SUPERHEATER SINCE THE GAS IS CLEAN.
2. EVAPORATOR SUBJECTED TO LOWER TEMPERATURE AND CLEANER GAS. METAL TEMPERATURE BELOW 400°C .
3. THE HIGHER THE STEAM CYCLE EFFICIENCY THE BETTER. CAN USE SPECIAL MATERIALS (INCONEL, STAINLESS) TO USE ADVANCED STEAM CYCLES TO REACH EFFICIENCIES CLOSE TO 50%.
4. TO MINIMIZE THE ENERGY LOSS TO THE STACK ($E7$) WE MUST LOWER THE EXHAUST GAS TEMPERATURE. USING A CONDENSER HEAT EXCHANGER (CHX) - TEFLON COATED TUBES IN COLDER SECTION OF ECONOMIZER WE CAN INCREASE THE OVERALL EFFICIENCY LOWERING THE EXIT TEMPERATURE OF THE FLUE GAS.



ST POWER = $151.65 * 0.36164 = 54.84 \text{ MWe}$

TOTAL POWER OF PLANT = $46 + 54.84 = 100.84 \text{ MWe}$

OVERALL EFFIC. = $100.84 / (112 + 40 + 71) = 45.22\%$

PAID FUEL EFFIC. = $100.84 / (112 + 40) = 66.34\%$

NG EFFIC. = $(100.84 - 20.93) / (112 + 40) = 52.57\%$

WASTE EFFIC. = $20.93 / 71 = 29.48\%$

WASTE APPAR. POWER = $100.84 - (112 + 40) * 0.51$

$= 23.32 \text{ MWe}$ (23.12% of total)

WASTE APPAR. EFFIC. = $23.32 / 71 = 32.84\%$

**FULL MIXING
WITH CHX**

TRADITIONALLY, RECOVERING HEAT FROM EXHAUST GAS HAS BEEN ECONOMICALLY VIABLE ONLY AT TEMPERATURES ABOVE THE ACID DEW POINT OF THE GAS. BELOW THIS POINT, COLD-END CORROSION WAS INEVITABLE, RESULTING IN PREMATURE EQUIPMENT REPLACEMENT COSTS.

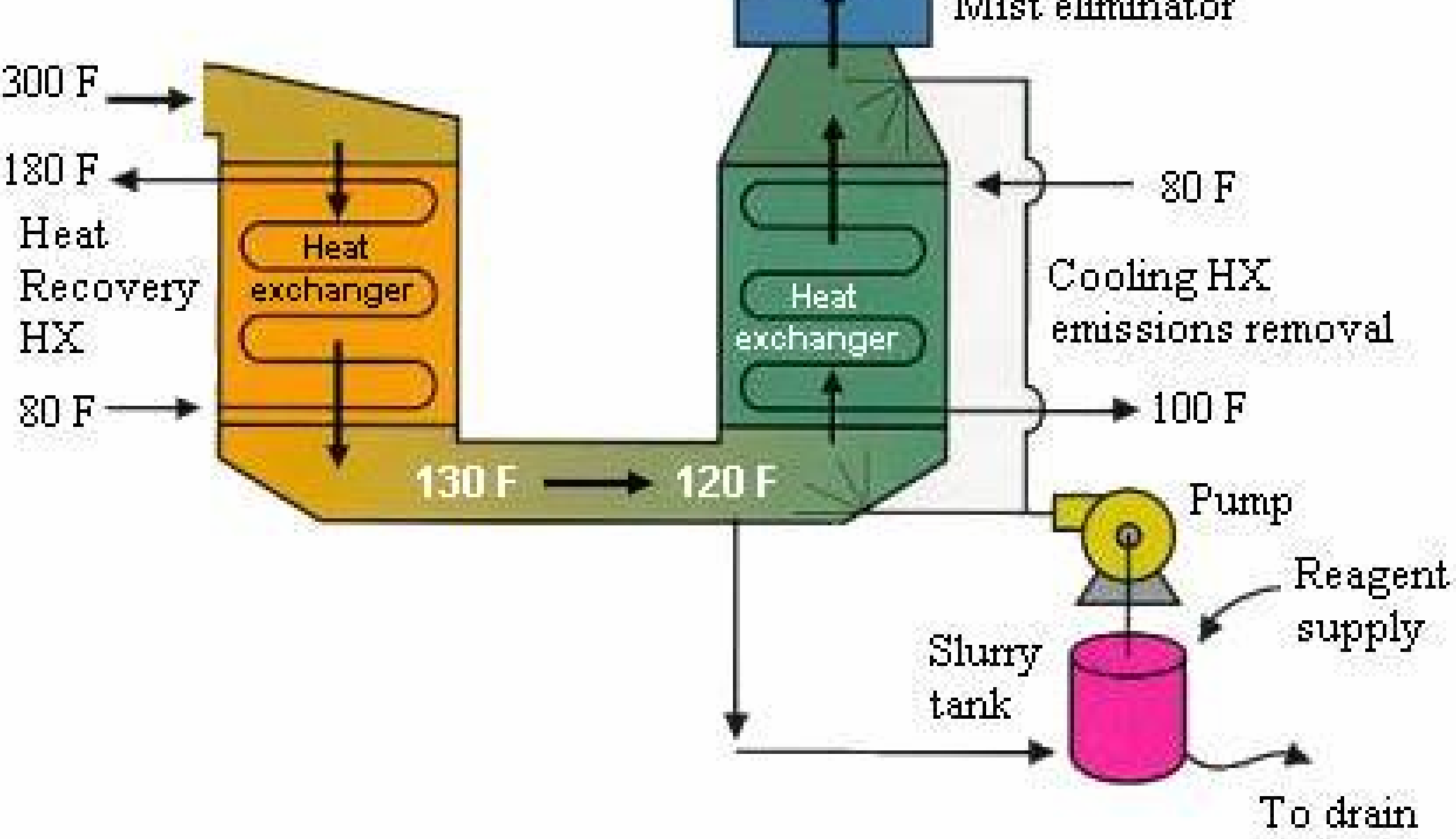
CONDENSING HEAT EXCHANGER

900 COMMERCE DRIVE, CLIFTON PARK, NY 12065

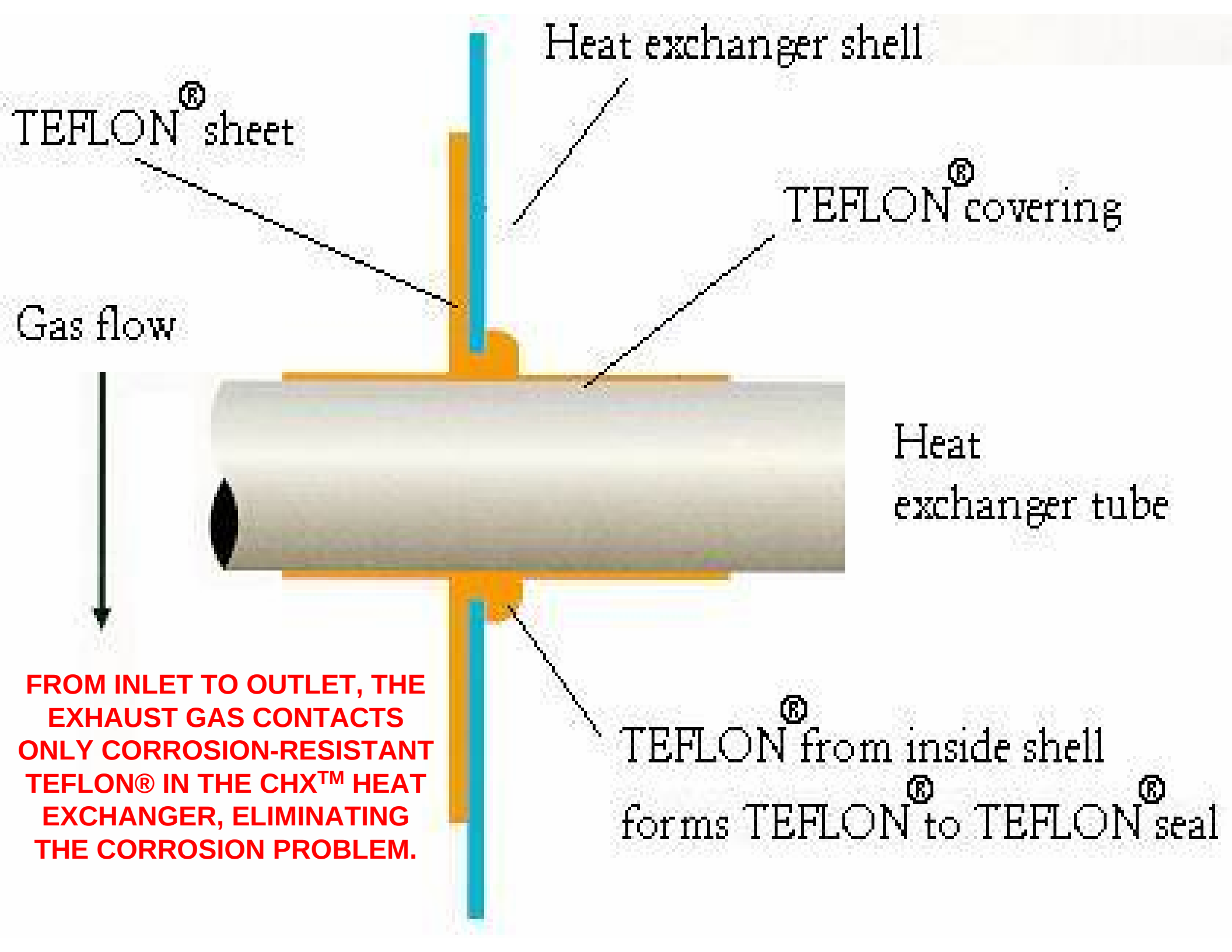
PHONE: (518) 877-8805 FAX: (518) 877-8686

E-MAIL: sps@capital.net

UNLIKE CONVENTIONAL ECONOMIZERS AND AIR HEATERS, WHICH MUST MAINTAIN TEMPERATURES ABOVE 300 DEGREES F (149 DEGREES C) TO PREVENT CONDENSATION, THE CHX™ SYSTEM THRIVES IN A CONDENSING ENVIRONMENT WHERE INLET WATER OR AIR TEMPERATURES ARE IDEALLY LESS THAN 80 DEGREES F (27 DEGREES C), AND EXHAUST GAS EXIT TEMPERATURES ARE ONLY 100 DEGREES TO 130 DEGREES F (38 DEGREES TO 54 DEGREES C).



SINCE THE HEAT EXCHANGER TUBE SURFACE AND THE EXHAUST AREA OF THE HEAT EXCHANGER ARE AT A TEMPERATURE WELL BELOW THE DEW POINT OF THE WATER VAPOR, A RAIN OF CONDENSATE IS PRODUCED THROUGH DROPWISE CONDENSATION OF THE WATER VAPOR. THIS CONDENSATE PASSES AROUND THE TUBE ARRAY, CARRYING PARTICULATES AND ACIDS WHICH HAVE BEEN SCRUBBED FROM THE GAS AND WASHED FROM THE TUBES.



Heat exchanger shell

TEFLON® sheet

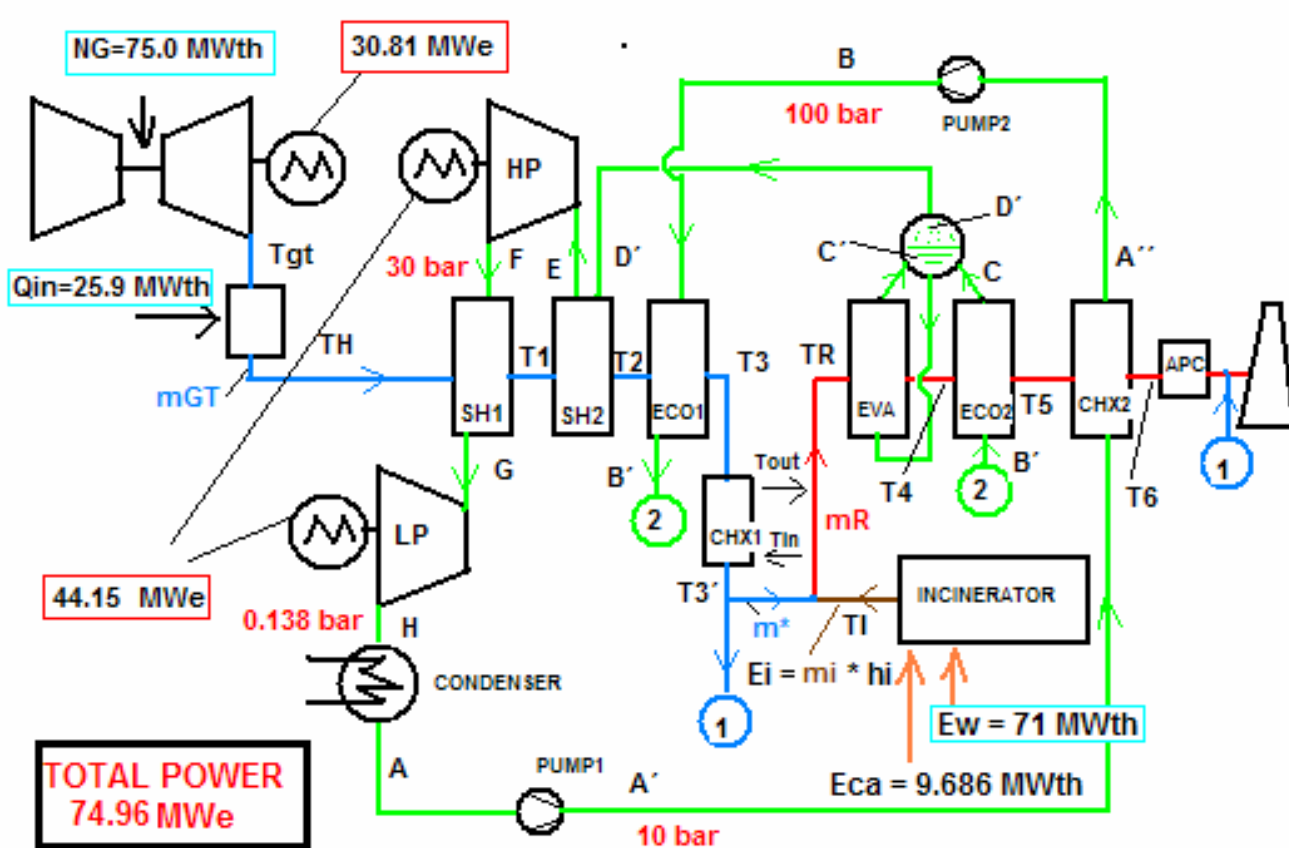
TEFLON® covering

Gas flow

Heat
exchanger tube

FROM INLET TO OUTLET, THE
EXHAUST GAS CONTACTS
ONLY CORROSION-RESISTANT
TEFLON® IN THE CHX™ HEAT
EXCHANGER, ELIMINATING
THE CORROSION PROBLEM.

TEFLON® from inside shell
forms TEFLON® to TEFLON® seal



GAS	WATER	PROCESS
T _{tg} = 455°C	TA = TA' = 52°C	mGT = 199.57 lbm/s
TH = 700°C	TA'' = TB = 105°C	m* = 33.542 lbm/s
T1 = 578°C	TB' = 194°C	mR = 185.28 lbm/s
T2 = 333°C	TC = 300°C	mi = 151.737 lbm/s
T3 = 205°C	TC' = 311°C	Tin = 16°C
T3' = 104°C	TD' = 311°C	Tout = 176°C
TR = 850°C	TE = 540°C	
T4 = 395°C	TF = 368°C	
T5 = 214°C	TG = 540°C	
T6 = 134°C	TH = 52°C	
TI = 1000°C		

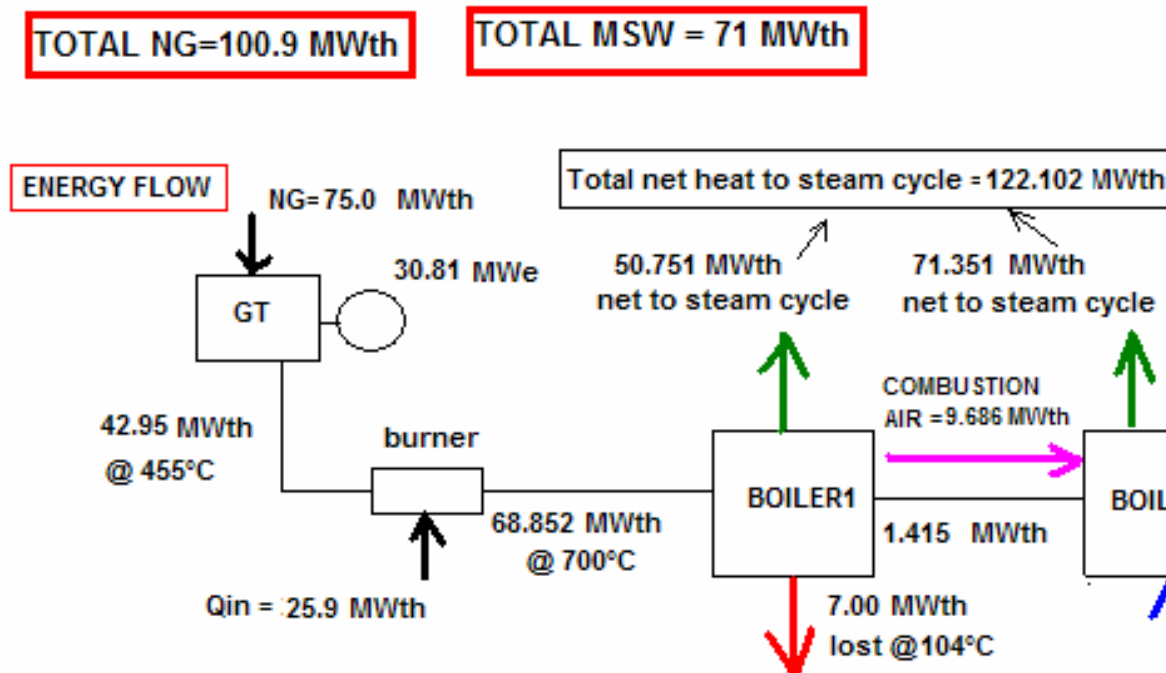
Extra energy in hot combustion air = 9.686 MWth
 STEAM CYCLE EFFIC. (NEGLECT PUMP WORK)=36.164%

WASTE CONTRIBUTION

BOILER 2 → ENERGY INPUT FROM MSW = 86.48%
 MSW CONTRIBUTION TO TOTAL POWER =
 $.8648 \times 71.351 \times 0.36164 = 22.31$ MWe or 29.8% OF TOTAL

ST POWER = $122.102 \times 0.36164 = 44.15$ MWe
 TOTAL POWER OF PLANT = $30.81 + 44.15 = 74.96$ MWe
 OVERALL EFFIC. = $74.96 / (75.0 + 25.9 + 71) = 43.6\%$
 PAID FUEL EFFIC. = $74.96 / (75.0 + 25.9) = 74.29\%$
 NG EFFIC. = $(74.96 - 22.31) / (75.0 + 25.9) = 52.18\%$
 WASTE EFFIC. = $22.31 / 71 = 31.42\%$
 WASTE APPAR. POWER = $74.96 - (75.0 + 25.9) \times 0.51 = 23.501$ MWe (31.35% total)
 WASTE APPAR. EFFIC. = $23.501 / 71 = 33.1\%$

SPLIT ECONOMIZER WITH CHX



SUMMARY

THERMAL POWER FROM WASTE = 71 MWth

STEAM CYCLE EFFICIENCY = 36%

SCHEME	TOTAL POWER (MW)	POWER FROM MSW		POWER FROM NG		OVERALL EFFICIENCY %	MSW EFFICIENCY %	NG EFFICIENCY %	MWh/ton	
		(MW)	%	(MW)	%				1850 7.7	2400 (Kcal/Kg) 10 (MJ/Kg)
BILBAO (ZABALGARBI)	98.06	21.30	21.7	76.76	78.3	44.0	30.0	50.50	0.64	0.84
FULL MIXING	100.84	20.93	20.8	79.91	79.2	45.2	29.50	52.57	0.63	0.82
SPLIT ECONOMIZER	74.96	22.31	29.8	52.65	70.2	43.6	31.4	52.18	0.68	0.88

USINAVERDE POWER UPGRADE PROJECT – UNIVERSITY OF BRASIL CAMPUS



2006/11/15

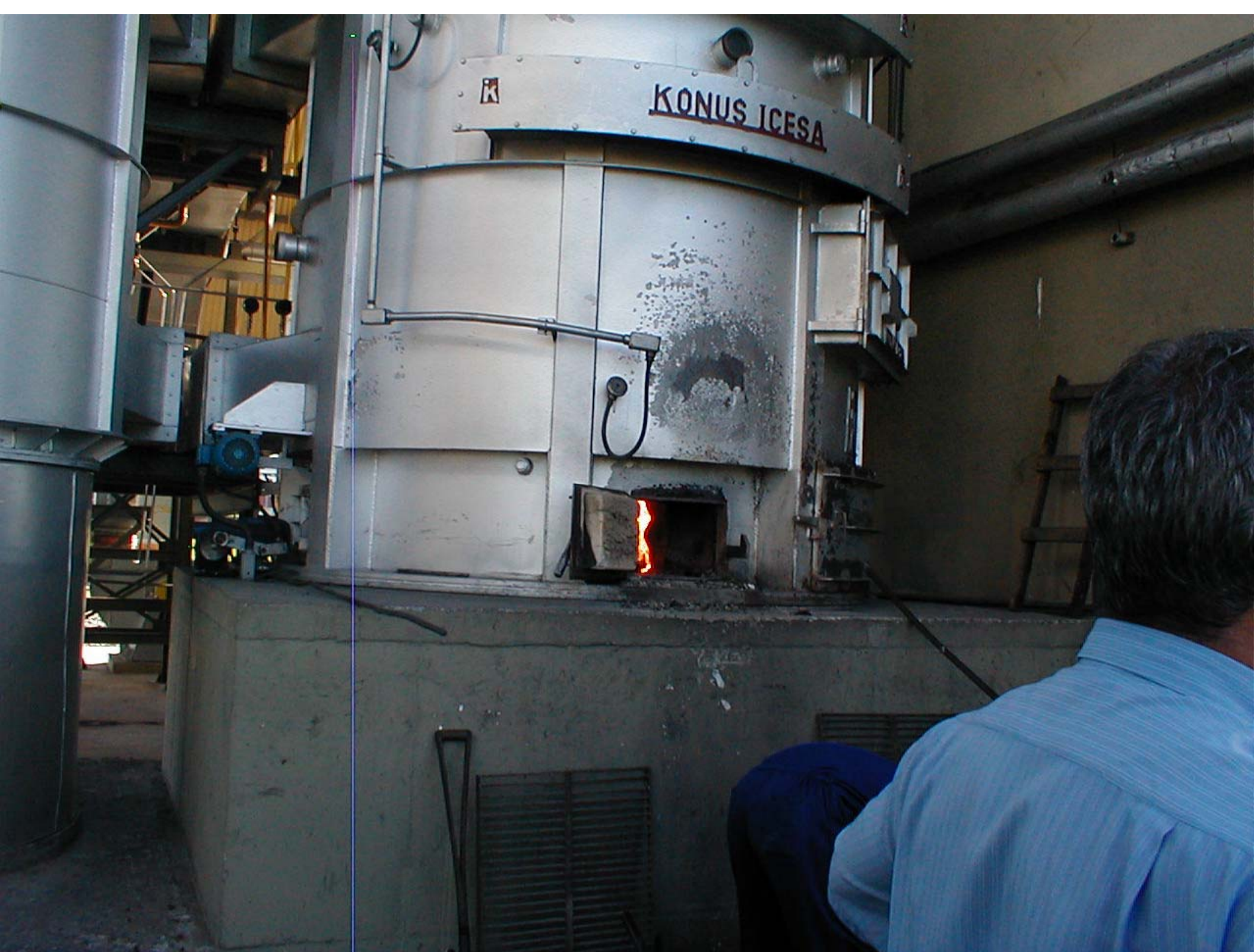
IVIG - Instituto Virtual Internacional de Mudanças Globais

Centro de Tecnologia, Bloco I, Sala 129 – CEP: 21945-970 – Rio de Janeiro – RJ

Telefax:(21) 2270-1586 – Site:www.Ivig.com.br







PROJECT DESCRIPTION

PARTNERS → COPPE, BR-DISTRIBUIDORA, PETROBRAS,
USINAVERDE.

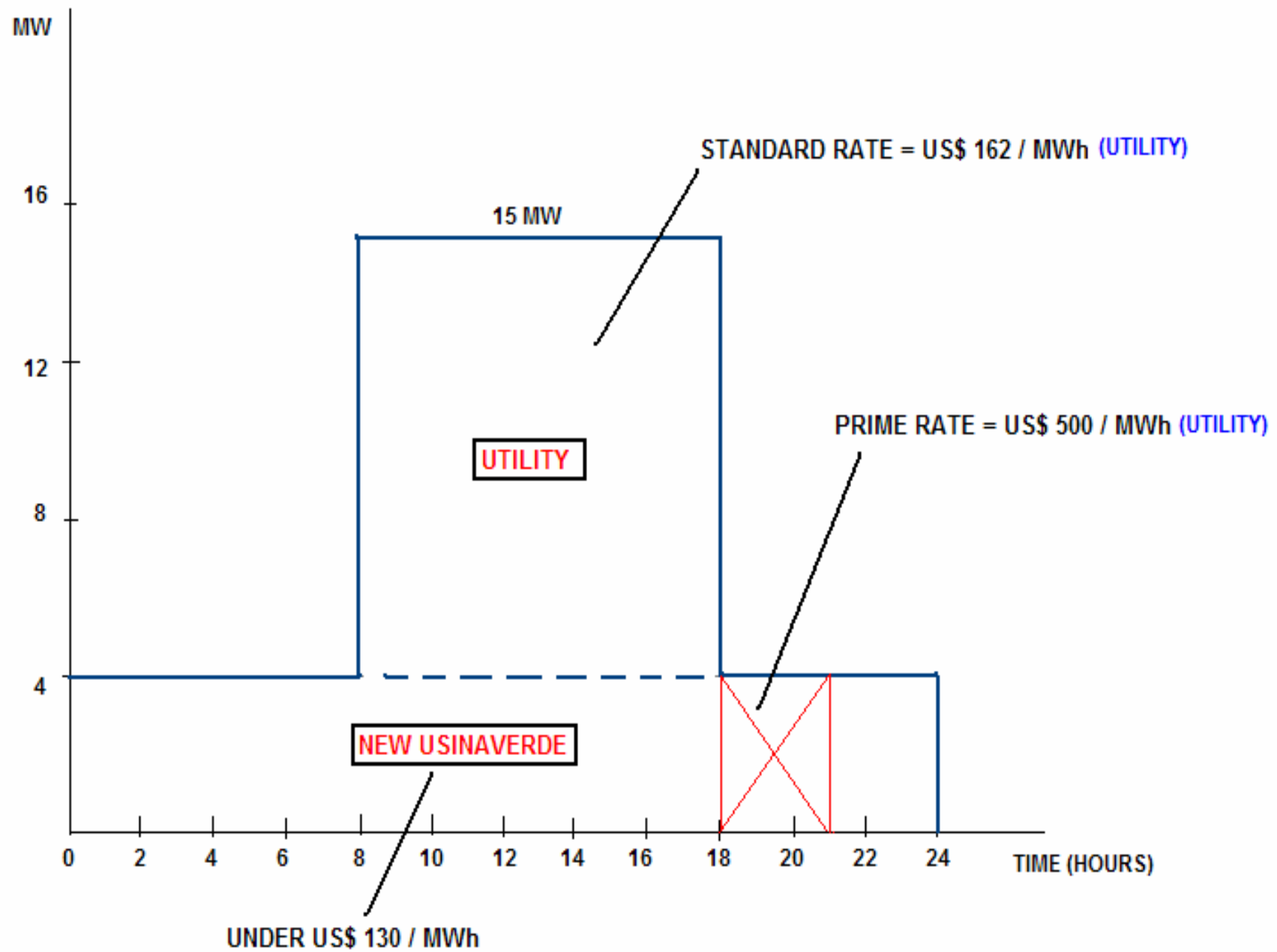
OBJECTIVE → GENERATE 4 MWe TO CAMPUS (BASE)

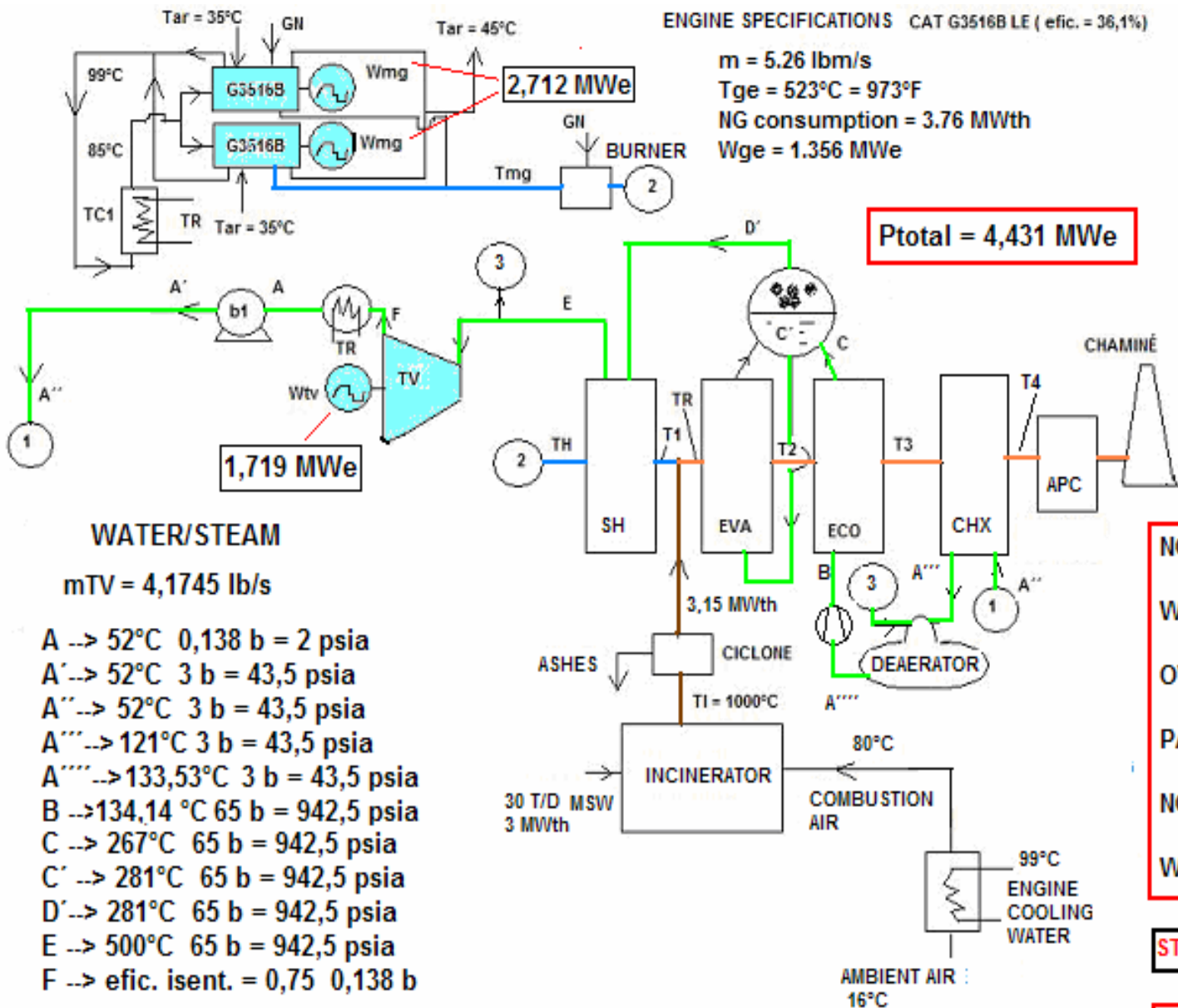
FINANCING → FINEP, BR

TOTAL TIME → 2 YEARS

WASTE EFFICIENCY → 0,74 MWh / TON (2.000 KCAL / KG)

UNIVERSITY OF BRASIL POWER CONSUMPTION - TYPICAL WEEK DAY





**USINAVEVERDE POWER
UPGRADE - RIO DE JANEIRO**