LONG TERM HHV DETERMINATION OF MUNICIPAL SOLID WASTE — A PRACTICAL APPROACH

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ABSTRACT

The determination of waste HHV over extended time periods is achievable by applying the concept of a specific steam correlation method developed from results of multiple boiler calorimetry tests. This correlation, used in conjunction with appropriate corrections for critical boiler operating parameters, provides a practical and reliable method of waste HHV determination. The theoretical basis, calculations and limitations of this HHV determination technique are presented and discussed.

INTRODUCTION

Waste processing capacity and energy recovery rate are the two primary production parameters used to gauge the performance of a resource recovery facility and are typically guaranteed on a long term basis. A capacity throughput guarantee is based on combusting a minimum quantity of waste having a specific "reference" higher heating value (HHV) and is typically expressed in terms of tons of waste processed per month or year, adjusted for the reference heating value. An energy recovery guarantee is based on producing a minimum amount of energy from waste at the same reference HHV and is typically expressed in kilowatt-hours/ reference ton or pounds of steam/reference ton of waste. The reference HHV is specified in conjunction with the waste processing or energy recovery guarantee due to the heterogeneous nature of waste and the large impact varying waste HHV has on these production parameters.

The determination of waste HHV is necessary to determine the difference between the actual waste HHV and the reference HHV for purposes of adjusting the actual production parameters to equivalent reference parameters which would have been obtained had waste of reference composition and HHV been processed. The performance of a facility can then be accurately assessed for purposes of:

(a) Demonstrating monthly or annual energy recovery and waste processing guarantees.

(b) Ensuring that the facility is being operated at the optimum capability and efficiency desired.

The purpose of this paper is to present a method for the determination of municipal solid waste HHV over extended time periods which is practical, accurate and economical to implement and also exhibits acceptable accuracy.

BACKGROUND OF HIGHER HEATING VALUE DETERMINATION

Conventional Fuels

The higher heating values of conventional fuels, such as coal, oil and gas are typically determined by regularly obtaining small representative samples of the fuel and performing an analysis for energy content (HHV) on a Btu/lb basis. This laboratory HHV determination is performed by utilizing a bomb calorimeter. The bomb calorimeter analysis consists of combusting typically a 1 gram sample of fuel in an airtight enclosure, surrounded by water. The heat generated from combusting the fuel is transferred to the water and measured by the temperature rise of the water jacket. The fuel HHV is then easily calculated.

Municipal Solid Waste

Municipal Solid Waste, with its inherent variability in composition and energy content (HHV), does not allow one to obtain a truly representative sample for laboratory analysis of HHV and thus leads to highly variable or questionable results when applying the conventional bomb calorimetry technique. Recognizing the difficulties with obtaining representative waste HHVs in a conventional laboratory calorimeter, the National Bureau of Standards embarked on the development of a larger calorimeter in the 1970s. Shortly thereafter it was determined that even with a larger calorimeter, representative waste HHV results were difficult to obtain. Because of these difficulties, the boiler-as-a-calorimeter test method gained wider acceptance and became the preferred method for waste HHV determination.

The boiler-as-a-calorimeter method is based on the similar concept of a laboratory calorimeter where all of the heat produced from the combusted refuse is measured across the boiler boundary, either as heat losses or recovered energy. The obvious and critical difference, however, is that instead of measuring several "representative" grams of refuse, one is literally measuring the heat released from tons of refuse. The boileras-a-calorimeter test method combines: (a) the heat loss method for boiler efficiency determination contained in the American Society of Mechanical Engineers (ASME) Power Test Codes (PTC) 4.1 and 33; (b) portions of the input-output method for boiler efficiency calculation contained in PTC-4.1; and (c) various equations and assumptions to combine the referenced documents into one cohesive test calculation method. The technical community within the resource recovery industry and the ASME have long recognized the need to develop a comprehensive, standardized Power Test Code for conducting and analyzing data from boiler calorimeter tests. This code, designated as PTC-34, Waste Combustors with Energy Recovery, is currently being developed by the ASME and in the near future will become the reference document for conducting



FIG. 1 BOILER AS A CALORIMETER TEST METHOD

tests and performing the required calculations for determining waste HHV.

The HHV determination performed in accordance with the boiler-as-a-calorimeter test method (refer to Fig. 1) is based on the premise that the HHV can be calculated by measuring the heat output of the boiler and translating it into heat input by determining one variable; namely boiler efficiency (i.e., heat output divided by boiler efficiency = heat input). The output is the product of the boiler steam flow and the energy required to generate and superheat the steam. The input is the product of the HHV and the measured quantity of the waste. The boiler efficiency is determined in accordance with the Heat Loss method in PTC-4.1/Reaffirmed 1979.

The test duration for a boiler calorimetry test is typically 8 hr. The determination of waste HHV for this relatively short period, in itself, poses a formidable challenge to the facility operator due to the costs associated with such a labor-intensive undertaking and the many data inputs required (refer to Fig. 2). For this reason, the boiler-as-a-calorimeter test method is primarily used during a facility's initial performance test and is not a practical option for long-term determination of HHV.

ALTERNATIVE METHODS FOR LONG TERM HHV DETERMINATION OF MSW

Any practical method considered for long term waste HHV determination must make a compromise between



- **1** Refuse Feed
- **2** Feedwater Flow
- **3** Steam Flow
- **4** Steam Temperature
- **5** Steam Pressure
- 6 Feedwater Temperature
- 7 Feedwater Pressure
- 8 Continuous Blowdown Flow
- 9 Drum Pressure
- 10 Ambient Dry Bulb Temp.
- 12 Ambient Wet Bulb Temp.
- 13 Overfire Air Flow
- **14 Combustion Air Flow**
- 15 CO2,O2,H20 in Flue Gas
- 16 Flue Gas Flow
- 17 Economizer Exit Temperature
- 18 Residue Rate, Moisture, and Unburned Combustibles
- **19 Ashpit Water Temperature**

test accuracy and the time duration over which the HHV determination can be performed.

Three methods typically considered for long term waste HHV determination are the extrapolation method, the instrumentation method and the specific steam correlation method. Each of these methods represents a different compromise between test accuracy and the duration over which boiler data and corresponding heat output data is measured.

Extrapolation Method

This method consists of conducting multiple 8-hr boiler calorimetry tests in accordance with the general guidelines outlined in PTC-4.1. and PTC-33. The 8-hr tests are performed at regularly specified intervals (i.e., daily, weekly, and monthly) and the results are extrapolated to provide an estimate of the waste HHV combusted during the remaining portion of that interval in which the HHV was not actually measured. For example, an 8-hr boiler calorimetry test might be performed weekly and the HHV determined for that 8-hr period would be assumed or "extrapolated" to be the same HHV for the entire week. The extrapolation method maintains the accuracy associated with the boiler-as-a-calorimeter method, but compromises on the duration of the analysis period. Since the actual HHV analysis period is less than 5% of the total time interval, significant errors in the HHV determination can arise. The concern over the impact of the limited analysis period becomes even more evident when one considers the heterogeneous nature of the waste and its

correspondingly widely fluctuating HHVs. The error could be reduced by conducting tests more frequently, but the large expense and manpower needs associated with performing an ASME quality boiler calorimetry test (e.g., \$10,000 or more) become prohibitive factors.

For these reasons, the extrapolation method is considered impractical for use in the determination of refuse HHV over extended time periods.

Instrumentation Method

The instrumentation method consists of implementing a system which continuously acquires the data necessary for determining the major boiler efficiency heat losses. Therefore, the duration of the analysis period is not compromised.

The continuous determination of the major losses and credits requires the acquisition of many data inputs, some of which can only be accurately and reliably obtained using prohibitively expensive manual test methods; namely, the determination of flue gas flow, flue gas moisture and residue unburned combustibles. Currently, the state of the art for instruments which continuously determine flue gas moisture and flue gas flow is not sufficiently developed or commercially proven where they may be deemed reliable or accurate in a harsh flue gas environment. Therefore, implementation of the instrumentation method for long term refuse HHV determination is considered unachievable.

Specific Steam Correlation Method

Recognizing that the extrapolation and instrumentation methods rely on techniques which are either unac-



ceptable or unachievable from the standpoint of accuracy, cost, manpower and available instrumentation, a third method — the specific steam correlation method — has been investigated. The specific steam correlation method is based on continuous data input for the major boiler parameters and therefore no compromise is made in the duration of the analysis period. Furthermore, the method uses inputs from standard commercially proven plant instruments and therefore can be easily implemented.

The specific steam correlation method is based on a concept which establishes a relationship between waste HHV and two regularly monitored production parameters: namely, waste processed and steam produced. The ratio of these two production parameters (pounds of steam produced per pound of waste processed) is termed the specific steam ratio (SSR). The SSRs and corresponding waste HHVs obtained from multiple boiler calorimetry tests are used to establish a correlation for use in determining the waste HHV on a long-term basis (refer to Fig. 3).

The HHV determination for a given interval is accomplished by: (a) measuring the waste processed and steam produced for the entire interval; (b) calculating the SSR; and (c) obtaining the corresponding HHV from the correlation.

DEVELOPMENT OF THE SPECIFIC STEAM CORRELATION METHOD

Ogden Martin Systems has (OMS) conducted multiple boiler calorimetry tests at 11 OMS facilities as part of acceptance test programs. The results of 37 such tests are presented in Table 1.

To enable a meaningful comparison of the test results among the facilities, differences in boiler operating conditions from facility to facility were adjusted to a common or normalized basis. It was realized during this early stage of development that a general or overall correlation based on a large number of points would be more statistically valid than a correlation from an individual facility based on three points. OMS proceeded to develop the specific steam correlation method for long term determination of refuse HHV using this overall correlation as follows:

(a) Data Analysis: Data and results from nine facility acceptance test programs, performed in accordance with applicable ASME Performance Test Codes were analyzed to develop the overall correlation of specific steam ratios to HHVs.

(b) Operating Parameter Adjustments: A mechanism was established to adjust the HHVs obtained from the correlation for differences between the baseline operating parameters on which the correlation is based and those measured during the desired HHV determination period.

(c) Validation: The accuracy of the method was determined empirically using data and results of nine ASME boiler calorimetry tests. In addition, an analytical model was used to compare the test results to theoretical results.

(d) Implementation: The continuous data acquisition requirements were established, along with corrections, for obvious operational influences affecting the method, such as boiler downtime and auxiliary fuel usage.

Data Analysis

To normalize the test results among the facilities, each test specific steam ratio — SSR — (lb steam/lb refuse) was converted to an equivalent specific steam output — SSO — (Btu/lb refuse). In addition, the test results were adjusted to the most prevalent boiler operating conditions. The baseline values are listed below:

economizer exit gas temp = 430° F

ambient air temp = 80° F

air preheat heat credit = 0 Btu/lb

excess air = 90%

unburned carbon in residue = 2.25%

unaccounted heat loss = 0.25%

The overall specific steam correlation was developed from the results of 28 of 37 of these boiler calorimetry tests, with pertinent data shown in Table 2. The test results were transformed into a mathematical equation

TABLE 1

				Specific	
	Tast	Steam	Refuse	Steam	Refuse
	Date	Flow	Flow	Batio	HHV
Plant		(LB/HR)	(LB/HR)	(LBstm/LBref)	(BTU/LBref)
Babyion, NY	21-Feb-89	173,087	57,889	2.99	4750
Babylon, NY	22-Feb-89	173,719	60,530	2.87	4693
Babylon, NY	23-Feb-89	178,161	56,186	3.17	4871
Babylon, NY	27-Feb-89	176,204	48,413	3.64	5556
Bristol, Ct.	03-Feb-88	135,670	51,639	2.81	4687
Bristol, Ct.	11-Feb-88	141,790	48,612	2.93	4851
Fairfax, Va.	21-May-90	731,216	274,458	2.64	4331
Fairlax, Va.	23-May-90	736,496	241,220	3.03	4740
Fairtax, Va.	24-May-90	740,008	255,890	2.87	4459
Haverhill, Ma.	25-May-89	394,609	126,273	3.13	5197
Haverhill, Ma.	26-May-89	416,523	146,924	2.83	5099
Haverhill, Ma.	27-May-89	416,491	140,573	2.96	5157
Huntsville, Al.	25-Jun-90	99,270	25,728	3.57	5095
Huntsville, Al.	26-Jun-90	95,707	21,784	4.24	5783
Huntsville, Al.	09-Jul-90	103,430	32,582	2.92	4163
Huntsville, Al.	10-Jul-90	99,133	27,772	3.28	4749
Huntsville, Al.	14-Jul-90	102,086	26,089	3.68	5345
Indianapolis	10-Nov-88	675,778	226,488	2.98	4180
Kent County, Mi.	04-Jan-90	156,460	47,834	3.27	5560
Kent County, Mi.	05-Jan-90	161,718	43,394	3.73	5979
Kent County, Mi.	08-Jan-90	157,040	50,168	3.13	5366
Kent County, Mi.	10-Jan-90	155,449	46,405	3.35	5558
Lake County, Fl.	15-Jan-91	137,501	45,015	3.05	5041
Lake County, Fl.	16-Jan-91	134,234	47,495	2.83	4704
Lake County, Fl.	17-Jan-91	134,864	46,898	2.88	4827
Lancaster, Pa.	26-Mar-91	306,859	95,154	3.22	5232
Lancaster, Pa	28-Mar-91	315,365	100,345	3.14	5058
Lancaster, Pa.	02-Apr-91	312,225	104,938	2.98	5199
Lancaster, Pa	04-Apr-91	310,416	97,818	3.17	5362
Lancaster, Pa.	05-Apr-91	310,115	99,388	3.12	5252
Pasco County, FL	15-Apr-91	274,717	95,220	2.89	4574
Pasco County, Fl.	17-Apr-91	282,051	94,480	2.99	4742
Pasco County, Fl.	19-Apr-91	279,919	95,101	2.94	4640
Pasco County, FL	22-Apr-91	270,865	101,130	2.68	4338
Stanislaus, Ca.	13-Dec-88	202,170	73,064	2.77	4657
Stankslaus, Ca.	16-Dec-88	208,940	68,658	3.04	4919
Stanislaus, Ca.	06-Jan-89	202,131	77,285	2.62	4474

by performing a linear regression analysis. Results from the remaining nine tests, not used in generating the original correlation, were then used for validation purposes by comparing the HHV results determined by the boiler-as-a-calorimeter method to the HHVs determined by the specific steam correlation method, as is discussed later.

The specific steam output correlation (HHV versus Btu/lb refuse) based on the 28 boiler calorimetry tests is shown in Fig. 4(a). The specific steam output correla-

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Plant Babylon, NY					Ambient	Heated	Select and a local		Receiver	Specific 1	Refuse
Plant Babvion, NY		Feedwater	Steam	Steam	Air	₹.	Heat	Bacene	Bart Own	Steam	NHH
Plant Babvion, NY	Test	Temp	Temp	Prom	Temp	Temp	Credit	Air	Temp	Output	
Babyion. NY	Date	(F)	(F)	(PSIG)	(F)	(F)	(BTU/LB)	(%)	(F)	(BTU/LB)	(BTU/LB)
	21-Feb-89	241	700	641	53	225	182	64	406	3262	4779
Babylon, NY	22-Feb-89	244	703	637	47	232	193	85	420	3158	4678
Babylon, NY	23-Feb-89	245	700	639	44	236	215	83	412	3461	4833
Babylon, NY	27-Feb-89	246	705	640	48	241	259	78	425	3970	5601
Bristol, Ct.	03-Feb-88	243	830	848	32	244	250	98	421	3267	4680
Bristol, Ct.	11-Feb-88	238	828	850	40	204	206	16	419	3436	1614
Fairfax, Va.	21-May-90	246	825	867	70	1/2	242	86	421	3000	4414
Fairfax, Va.	23-May-90	247	819	867	62	275	236	103	405	3452	4823
Fairfax, Va.	24-May-90	248	820	870	63	282	221	16	404	3246	or 4542
Haverhill, Ma.	25-May-89	251	813	861	62	8	21	z	428	3683	5136
Haverhill, Ma.	26-May-89	250	831	855	80	95	20	104	457	3433	4834
Haverhill, Ma.	27-May-89	250	831	854	78	8	21	102	460	3593	5032
Huntsville, Al.	25-Jun-90	251	460	344	86	223	167	83	124	3500	5056
Huntsville, Al.	26-Jun-90	251	463	355	87	217	182	80	398	4066	STT2
Huntaville, Al.	09-Jul-90	255	476	348	85	180	\$	87	421	1162	4138
Huntsville, Al.	10-Jul-90	259	472	351	8	206	145	100	447	3273	4571
Huntsville, Al.	14-Jul-90	250	465	352	80	215	178	64	446	3634	5349
Indianapolis	10-Nov-88	243	708	512	S 8	294	232	100	452	2905	4104
Kent County, Mi.	04-Jan-90	249	836	879	58	186	179	82	452	3847	5557
Kent County, Mi.	05-Jan-90	250	832	887	49	252	296	82	451	4314	0609
Kent County, Mi.	08-Jan-90	244	840	892	3	175	171	86	458	3726	5320
Kent County, Mi.	10-Jan-90	246	828	881	52	180	181	8	450	3959	566 8
Lake County, Fl.	15-Jan-91	242	812	864	69	220	226	8	397	LSNE	5111
Lake County, Fl.	16-Jan-91	242	816	863	74	254	260	66	399	3178	4761
Lake County, Fl.	17-Jan-91	242	814	863	63	218	218	102	399	3282	4905
Stanialaus, Ca.	13-Dec-88	242	820	863	99	267	215	104	434	3316	5311
Stanialaus, Ca.	16-Dec-88	242	608	858	55	265	216	95	413	3528	5163
Stanislaus, Ca.	06-Jan-89	243	830	856	8	269	174	68	425	3111	5273









tion (btu/lb refuse) is also presented on a specific steam ratio basis (lb steam/lb refuse) in Fig. 4(b) for two common final steam conditions in the waste-to-energy industry; 830°F, 865 psia and 700°F, 670 psia. The correlation method is presented on a specific steam ratio basis (lb steam/lb refuse), since in this form, it is expressed in terms of measured parameters familiar to the facility operator.

A statistical analysis of the data indicates that: (a) the resulting r^2 value equals 0.95 (i.e., 1.0 representing a theoretically perfect correlation); and (b) the standard error of Y estimate (i.e., the HHV) equals 113 Btu/lb. This standard error, expressed as a percentage of a typical HHV of 5000 Btu/lb combusted at a mass-burn facility, would yield an overall HHV determination tolerance of less than 2.3%.

Operating Parameter Adjustments

The boiler operating parameters with the greatest impact on boiler efficiency consist of:

- (a) Economizer gas outlet temperature.
- (b) Ambient air temperature.

(c) Preheated combustion air temperature.

(d) Flue gas oxygen concentration (i.e., excess air/flue gas flow).

(e) Unburned carbon.

The relationship of each of these boiler operating parameters to boiler efficiency and waste HHV content was established by individually varying the parameters in a mathematical model which uses the heat loss formulas in PTC-4.1. Specifically, the following heat losses were evaluated:

(a) Heat loss due to heat in dry flue gas.

(b) Heat loss due to moisture in the air.

(c) Heat loss due to moisture in the fuel and combustion of hydrogen to form moisture.

(d) Heat loss due to unburned carbon.

Using these equations, calculations were performed varying the three temperature parameters in 10 deg. increments, the excess air parameter in 10% increments and the residue unburned carbon in 1% increments to yield the respective changes in boiler efficiency. The analysis was performed using a typical waste composition (ultimate analysis) corresponding to a waste HHV of 5000 Btu/lb. The HHV of 5000 Btu/ lb was selected as the baseline waste for the analysis since it represents the midpoint of the typical range of HHVs combusted at a mass-burn facility. The analysis yields the following relationships (refer to Fig. 5):

Economizer Exit Gas Temperature

A 10°F increase (decrease) in economizer exit gas temperature from the baseline economizer exit gas temperature equates to a 0.4% change in boiler efficiency and a corresponding increase (decrease) of 0.57% in the refuse Btu content obtained from the correlation.

Ambient Air Temperature

A 10°F increase (decrease) in ambient air temperature from the baseline ambient air temperature equates to a 0.5% change in boiler efficiency and a corresponding decrease (increase) of 0.71% in the refuse Btu content obtained from the correlation.

Preheated Combustion Air Temperature

For every 10°F rise in combustion air temperature across the steam coil airheaters (supplied by steam from outside the boiler boundary), the refuse higher heating value obtained from the correlation will decrease (increase) by 12 Btu/lb.

27

55

6

Excess Air

For every 10% increase (decrease) from the baseline excess air percentage, the boiler efficiency will change by 0.6% and the higher heating value obtained from the correlation will correspondingly increase (decrease) 0.86%.



FIG. 5(a) AMBIENT AIR TEMPERATURE VERSUS BOILER EFFICIENCY



FIG. 5(c) PREHEATED AIR TEMPERATURE VERSUS BOILER EFFICIENCY

Unburned Combustibles

For every 1% increase (decrease) from the baseline unburned combustible percentage, the boiler efficiency will change by 0.6% and the higher heating value obtained from the correlation will increase (decrease) 0.86%.

Validation

The validity of the specific steam correlation method was demonstrated empirically, using the results of nine boiler calorimetry tests and supported theoretically using an analytical model.

Empirical Validation

HHVs calculated using the boiler-as-a-calorimeter test method were compared to the HHVs determined using the correlation method. Data and results from the two most recent boiler calorimetry test programs were used. The data for these test programs are pre-



FIG. 5(b) ECON OUTLET TEMP VERSUS BOILER EFFICIENCY



FIG. 5(d) EXCESS AIR VERSUS BOILER EFFICIENCY

sented in Table 3 and the results of the comparison are included in Table 4. The HHVs determined by the correlation method averaged 0.2% lower for the four Pasco tests and 1.9% higher for the five Lancaster tests than HHVs calculated by the boiler-as-a-calorimeter test method. The agreement within $\pm 2.5\%$ of the more rigorous boiler-as-a-calorimeter method demonstrates the empirical validity.

Theoretical Validation

Typical waste compositions corresponding to HHVs ranging between the extremes (for mass-burn waste) of 2500 Btu/lb and 9000 Btu/lb were used to develop a theoretical model. Keeping boiler operating parameters at their constant baseline values, calculations were performed to determine the boiler efficiency and specific steam ratio (SSR) for each of the respective waste compositions. The results of the analysis are summarized in Table 5. A plot of the theoretical specific steam correlation results is included in Fig. 6. When the re-

Specific Refuse Roomize Am Heated Feedwaler Air Rand Ball Gas HHV Steam Steam Air Heat Steam Tat Temp Credit Air Temp Press Temp Output Temp Temp Date (F) 250 (F) (PSIG) (F) BTU/LB (BTU/Ib) (F) 258 (%) **(F)** (BTU/LB) 26-Mar-91 274 Lancaster, Pa 862 423 5232 5058 805 83 3597 28-Mar-91 250 863 252 805 82 248 82 3517 Lancaster, Pa. 824 5199 Lancaster, Pa. 02-Apr-91 250 862 72 151 112 447 85 3490

75

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82

81

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74

183

177

230

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226

862

861

878

873

872

873

TABLE 3 SPECIFIC STEAM CORRELATION TEST DATA FOR CORRELATION VALIDATION

TABLE 4 COMPARISON OF BOILER-AS-A-CALORIMETER RESULTS VERSUS CORRELATION PREDICTION

04-Apr-91

05-Apr-91

15-Apr-91

17-Apr-91

19-Apr-91

22-Apr-91

250

250

294

299

296

297

816

812

811

840

823

830

		Measured	Predicted			RMS
		HHV	HHV	Difference	Difference,	Difference
Plant	Date	(BTU/LBr)	(BTU/LBr)	(BTU/LBr)	(%)	(%)
Lancaster	26-Mar-91	5232	5115	116	2.3	
Lancaster	28-Mar-91	5058	5003	55	1.1	12:20
Lancaster	02-Apr-91	5199	4991	208	4.2	2 64 54 85
Lancaster	04-Apr-91	5362	5269	93	1.8	
Lancaster	05-Apr-91	5252	5174	78	1.5	2.4
Pasco	15-Apr-91	4574	4640	-66	-1.4	-1 T T
Pasco	17-Apr-91	4742	4769	-27	-0.6	1.1.1
Pasco	19-Apr-91	4640	4686	-46	-1.0	
Pasco	22-Apr-91	4338	4334	5	0.1	0.0

sults of the 37 boiler calorimetry tests are also plotted on Fig. 6, it can be seen that there is excellent agreement between the theoretical model results and boiler calorimetry test results.

Implementation

Plant

Lancaster, Pa.

Lancaster, Pa.

oo County, Fl

co County, Pl.

Pasco County, Fl.

Pasco County, FL

Ogden Martin Systems (OMS) has implemented the specific steam correlation method at all OMS operating facilities. A specific steam correlation summary sheet, containing the data inputs, calculations and adjustments performed as part of this method is included in Table 6. The HHV determination using the specific steam correlation concept is accomplished as follows:

Step 1: Acquiring and Inputting Data

The following process parameters are used as data inputs:

Monthly Totals:

- (a) Refuse Throughput.
- (b) Boiler steam flow.
- (c) Auxiliary fuel usage.

Monthly Averages:

- (a) Boiler steam temperature.
- (b) Boiler steam pressure.
- (c) Boiler feedwater temperature.
- (d) Boiler economizer exit gas temperature.
- (e) Boiler heated combustion air temperature.

(f) Ambient air temperature.

160

154

150

159 153

147

(g) Boiler flue gas oxygen content O_2 .

91

89

90

87

89

90

450

445

455

447

445

452

3620

3229

3331

3266

2998

5362

5252

4574

4742

4640

4338

Step 2: Determining of Refuse Throughput

It is recommended that the minimum HHV determination interval consist of a one month period. This recommended period stems from the need at most facilities to rely on the truck scale weights with adjustments for pit volume difference for determining the quantity of waste combusted.

Step 3: Converting the Measured O₂ to Excess Air The measured O₂ concentration on a "dry" volumetric basis is used to calculate the excess air.

Step 4: Calculating the Average Hourly Heat **Output for the Month**

The heat output is based on the measured flow using the permanent plant feedwater or main steam flow elements. Each respective facility must be evaluated on an individual basis to include any additional output streams (if existing) into the heat output determination. In most cases the continuous blowdown flow can be assumed as having a negligible contribution to heat output.

The as-measured steam flow is normalized to reference steam temperature and pressure and feedwater temperature. This is accomplished by multiplying the measured steam flow by the ratio of the actual feedwater to steam enthalpy difference to reference feedwater to steam enthalpy difference.

Step 5: Adjusting the Weekly Heat Output for Any Auxiliary Fuel Fired

Steam flow generated from auxiliary fuel usage is calculated using the measured average fuel flow, a boiler efficiency corresponding to the fuel being fired and the energy required to generate a pound of steam. The calculated steam flow attributed to auxiliary fuel firing is then subtracted from the total average steam flow to obtain the average steam flow generated from waste alone.

	Ratio)
	Steam
RESULTS	Specific
EL	, and
AL MOI	fficiency
ALYTIC	Boiler E
5 AN	Versus
TABLE	Composition
	Waste

									9500		4.26	1.50	49.90	7.22	33.97	3.00	0.15	100.00		12.034	13.960	13.454		11.096	0.307	8.489	3.470	24.162	75.030		5.350		4.923	
									8000		4.07	3.20	47.62	6.90	35.22	2.65	0.14	100.00		12.068	13.105	12.607		11.041	0.304	8.003	3.470	24.418	75.582		5.058	100	4.664	
									9200		4.04	5.00	45.72	6.59	36.20	2.33	0.12	100.00		11.325	12.433	11.940		11.800	0.302	9.182	3.470	24.754	75.246	and the second	4.756		4.405	
									9009		5.06	6.00	43.48	6.25	36.20	2.03	0.10	100.00		10.606	11.686	11.208		11.768	0.301	9.444	3.470	24.903	75.017		4.463		4.146	
									7500		7.90	7.00	41.18	5.90	36.10	1.75	0.09	100.00		9.862	10.933	10.470		11.726	0.299	9.728	3.470	25.223	14.777		4.171		3.667	
									7000		9.95	9.00	36.73	5.54	35.20	1.50	0.08	100.00		9.192	10.166	9.731		11.706	0.298	10.210	3.470	25.664	74.316		3.868		3.628	
									6500		12.21	11.00	36.25	5.10	34.05	1.25	0.06	100.00		8.508	9.497	9.045		11.609	0.297	10.767	3.470	26.223	111.61		3.566		3.369	
									0009		14.53	13.00	33.77	4.01	32.78	1.06	0.05	100.00		7.828	8.766	8.339		11.674	0.296	11.398	3.470	26.838	73.162		3.264		3.110	.1
									\$500		16.57	16.00	31.13	4.43	30.96	0.87	0.04	100.00		7.152	8.080	7.629		11.652	0.295	12.346	3.470	27.763	12.237		2.954		2.850	
									2000		20.26	17.50	20.51	4.04	28.99	0.67	£0.0	100.00		6.487	7.369	6.929		11.640	0.294	13.096	3.470	28.500	71.500		2.650		2.591	
									4500		21.19	22.40	25.74	3.64	26.43	0.50	0.02	100.00		5.830	6.694	6.232		11.633	0.294	14.911	3.470	30.308	69.692		2.332		2.650	
_		_	_						4000		22.44	27.20	25.92	3.24	23.70	0.48	0.02	100.00		5.177	6.020	5.536	-	11.626	0.294	17.149	3.470	32.539	67.461		2.007		2.073	
		650 TPD)							3500		23.81	32.00	20.07	2.04	20.07	0.39	0.02	100.00	11 11 11 11 11 11 11 11 11 11 11 11 11	4.526	5.347	4.041		11.619	0.293	20.027	3.470	35.409	64.591	(SNC	1.601	(S)	1.014	
		QUIV. TO	65ps i a						0000		23.49	30.50	17.23	2.43	18.03	16.0	0.01	100.00		3.870	4.686	4.142	X	11.598	0.293	24.519	3.470	39.000	60.120	NSIDERATI(1.341	SIDERATION	1.555	
	DO LB/HR	5 LO/HR (E	830°F, 8	250°F	430°F	80°F	B0 ⁴ F	106	2500		23.27	45.00	14.38	2.03	15.06	0.25	0.01	100.00		3.230	4.040	3.457		11.617	0.293	30.853	3.470	46.233	53.767	ER EFF COI	1.000	R EFF CON	1.329	
BASE CASE CONDITIONS:	STEAM PRODUCE 144,00	WASTE COMBUSTED 54,164	FINAL STEAM CONDITIONS	BOILER FEEDWATER INLET	ECONOMIZER EXIT GAS TEMP.	AMBIENT AIR TEMP.	PREMEATED AIR TEMP	EXCESS AIR	MASTE COMPOSITIONS/HHV (BTU/LB)	(LB/100 LB MASTE)	ASH & INERTS	MOI STURE	CARBON	MY DROG EN	OXYGEN	N I TROGEN	SULFUR	TOTAL	AIR & GAS WEIGHTS (LB/LB WASTE)	DRY AIR	WET GAS	DRY GAS	BOILER MEAT LOSSES (*)	-ORY GAS	-MOIST. H2/H20 IN FUEL	-MOISTURE IN AIR	-FIXED LOSSES	TOTAL LOSS	BOILER EFFICIENCY (%)	SPECIFIC STEAM RATIO (WITH BOIL	(LB STEAM/LB REFUSE)	SPECIFIC STEAM RATIO (W/O BOILE	(LB STEAM/LB REFUSE	



FIG. 6 COMPARISON OF THEORETICAL VERSUS TEST $(Tg14 = 430^{\circ}F, Ta8 = 80^{\circ}F, 0 HtCr, 90\% Xs, and Luac$ = 0.25)

TABLE 6 MONTHLY HHV CALCULATION SHEET

Month:

SAMPLE

		MONTHLY	REFERENCE
DATA INPUTS	Units	VALUE	VALUES
Refuse Throughput	tons	4375	
Boiler 1 Steam Flow	kib	12862	
Boiler 2 Steam Flow	klb	12862	
Boiler 1 Stm Temp (Avg)	٩F	830	830 °F
Boiler 2 Stm Temp (Avg)	٩F	830	
Boiler 1 Stm Press (Avg)	psig	850	850 ps
Boiler 2 Stm Press (Avg)	psig	850	
Boiler 1 FW Temp (Avg)	°F	250	250 °F
Boiler 2 FW Temp (Avg)	٩F	250	
Boiler 1 Econ Exit Gas Temp (Avg)	٩F	430	430 °F
Boiler 2 Econ Exit Gas Temp (Avg)	٩F	430	
Boiler 1 Heated Comb Air Temp (Avg)	°F	80	80 °F
Boiler 2 Heated Comb Air Temp (Avg)	٩F	80	
Ambient Air Temp (Avg)	°F	80	80 °F
Boiler 1 Econ Exit O2 (Avg)	9%	8.7	
Boiler 2 Econ Exit O2 (Avg)	9%	8.7	
Aux Fuel Usage - Natural Gas	kcfm	0	1000 B

ENTHALPIES		
Main Steam	BTU/16	1414
Feedwater	BTU/Ib	221

96

BTU

lbs

lbs

BTU/Ib

1414	BTU/Ib
221	BTU/Ib

1000 BTU/CFM

830 °F

850 psig

250 °F

430 °F

80 °F

80 °F

90	%	12	
			_

85% Efficiency

Step 6: Calculating the Specific Steam Ratio

The specific steam ratio is determined by dividing the monthly quantity of steam generated by the total monthly quantity of waste combusted. Note that the steam quantity used in this Step 6 has been adjusted per Steps 4 and 5. The monthly processed tonnages are determined by using the waste quantity as measured by the calibrated truck scales and adjusted for changes in volume of waste in the pit between the beginning and ending of the monthly period.

Step 7: Determining the HHV from the Correlation (i.e., Using the Equation Determined from the Regression Analysis)

The unadjusted HHV is obtained from the specific steam correlation using the specific steam ratio determined in Step 6.

Step 8: Adjusting the Raw HHV Obtained from Step 7 for:

(a) Monthly average economizer exit gas temperature.

- (b) Monthly average combustion air temperature.
- (c) Monthly average ambient air temperature.
- (d) Monthly average excess air percentage.

ADJUSTMENTS		
Econ Gas Temp	BTU/Ib	0
Heated Combustion Air Temp	BTU/Ib	0
Ambient Air Temp	BTU/Ib	0
Excess Air	BTU/Ib	0

Factor]
0.57	% raw HHV/10*F
-12	BTU/D/10*F
0.71	% raw HHV/10°F
0.86	% raw HHV/%

Sub-total of adjustments BTU/Ib

Net HHV

CALCULATIONS

% Excess Air from %O2

Total Bir Steam Ht Output

HHV Raw Database Curve

Ref Stm Prod due to Aux Gas

Reference Total Stm Produced

Specific Stm Ratio (lb stm/lb refuse)

5000 BTU/b

0

90

3.07E+10

0.00E+00

2.57E+07

2.94

5000

DISCUSSION

The many factors which impact the HHV determination using the specific steam ratio method can be categorized into two distinct groups:

(a) Boiler operating parameters.

(b) Composition of the waste itself.

The influence of the major boiler operating parameters on boiler efficiency and in turn on HHV determination has been addressed by incorporating an adjustment mechanism. This adjustment mechanism accounts for the impact of these parameters on the specific steam ratio, as previously outlined in the section on Operating Parameter Adjustments.

The influence of varying waste composition on HHV determination is also accounted for by the specific steam correlation. This is achieved because implicit within the correlation are boiler efficiencies which reflect the effects of waste composition. For example, a boiler efficiency resulting from firing a lower waste HHV is less than a boiler efficiency resulting from firing a higher waste HHV. The higher waste moisture content that is typically present in a low waste HHV requires more energy from the combustion process in order to vaporize the larger quantity of water present (i.e., more latent heat of vaporization), thus resulting in a lower boiler efficiency. This waste composition

(e) Monthly average of residue unburned combustible.

Adjustments to the HHV for operating parameters are calculated by determining the deviations between the weekly average parameters and the reference "baseline" values for which the correlation was based. The relationships established in the theoretical model for operating parameter deviations are applied to determine the required adjustments to the unadjusted HHV obtained in Step 7.



FIG. 7 THEORETICAL HHV VERSUS SSR WITH AND WITHOUT ADJUSTMENT FOR BOILER EFFICIENCY

impact on boiler efficiency and on the specific steam ratio is illustrated in Fig. 7. One of the two correlations in Fig. 7 includes the impact of waste composition on the correlation. In this case, the boiler efficiency corresponding to 6000 Btu/lb waste equals 73.2%, while the boiler efficiency corresponding to 4000 Btu/ lb is equal to 67.5%, i.e., a boiler efficiency change of 5.7% results from a 2000 Btu/lb change in HHV, assuming identical boiler operating parameters (i.e., excess air, flue gas exit temperature, etc). If this influence of waste composition was not inherently incorporated into the correlation (e.g., if the boiler efficiency was incorrectly assumed to be constant over the HHV range), then a substantially different correlation arises (as illustrated in Fig. 7 by the second correlation). As indicated, the variation in boiler efficiency due to waste composition significantly impacts the correlation and corresponding HHV determination.

Waste moisture content has the most significant impact on boiler efficiency, but influences from other important waste composition constituents such as the carbon to hydrogen ratio, inerts fraction, carbon to oxygen ratio, etc. also exist. However, as evidenced by the excellent correlation, the collective influence of these other waste constituents can be deemed negligible.

CONCLUSIONS

The determination of municipal solid waste HHV, over extended time periods, is achievable using the specific steam correlation method. The test results of 37 boiler calorimetry tests at 11 different facilities, were used to develop the method.

The method is derived from recognized ASME Power Test Code methods and calculations and is, in essence, an extension of the theoretical basis of the boiler-as-a-calorimeter method. The method has demonstrated an accuracy of better than $\pm 2.5\%$.

In addition to having a sound theoretical basis, the method is practical and can easily be implemented at minimal cost using data typically available from conventional facility instrumentation.